



# Discrete/Distinct Element Method

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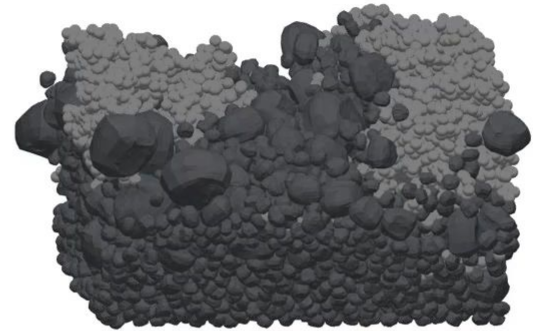
# Outline

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1. Introduction to DEM
2. Historical Perspective
3. General Principles
4. Governing Equations
5. Hand-calculation Example
6. Numerical example
7. Applications of DEM

## What is the Discrete Element Method (DEM)?

- Model of discrete matter
- Material is composed of several discrete particles
  - Particles can have different shapes and sizes
- Materials that can be simulated with DEM
  - Granular matter (e.g. sediment)
  - Powder
  - Granular flows and blocky masses (e.g. rockslides)
  - Bulk materials in storage
  - Liquids and solutions





# Introduction

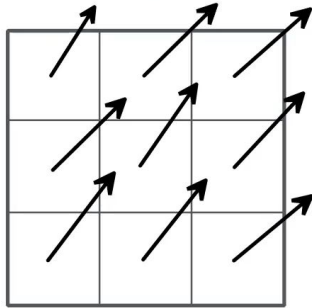
When it comes to simulating particulate systems, two main modeling approaches can be identified –continuum (Eulerian) and discrete (Lagrangian).

<i>Continuum</i>	<i>Discrete</i>
Continuous systems	Discontinuous, granular media
Assumes granular substance fills the space it occupies	Models behavior of individual particles
Relates stresses and strains through constitutive equations	Overall system behavior results from individual particle interactions
Suitable when length scale of importance is higher	Good for investigating phenomena occurring at particle length scale



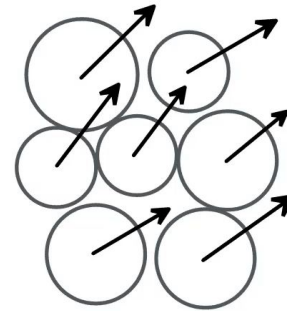
# Introduction

FEM (continuous)



- Occupies entire space
- Cannot calculate relative particle movements and rotations

vs. DEM (discrete)



- Discontinuous
- Can calculate particle displacements and rotations



# Introduction

## Advantages

- Performs well with **granular** and **discontinuous materials**
  - Good method for rock mechanics
- Enables visualization of position, velocity, force, stress, and strain networks
- Accounts for...
  - **Particle rotation**
  - **Time steps**
  - **Progressive failure**

## Disadvantages

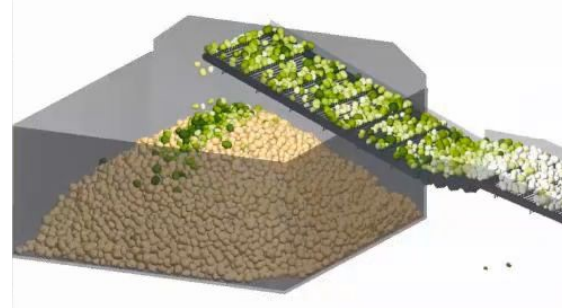
- Computationally intensive
- Limits number of particles and/or simulation length
- Difficult to capture...
  - Complex particle arrangements
  - Particle roughness
  - Particle breakage



# Introduction

## Industrial Applications

- Agriculture & food transport
- Oil and gas
- Mining
- Geomechanics
- Chemical engineering
- Civil engineering
- Pharmaceutical
- Powder metallurgy





# Historical Perspective

**Newton, 1697**

Theoretical principles

**Alder & Wainwright, 1956**

Molecular dynamics

**Peter Cundall, 1971**

Discrete Element Method  
“A computer model for simulating progressive large-scale movements in blocky rock systems”

**Williams, Hocking, & Mustoe, 1985**

DEM as generalized FEM

**Williams, Pande, & Beer, 1990**

Application to geomechanics  
*Numerical Methods in Rock Mechanics*

**Shi, 1992**

Discontinuous  
Deformation  
Analysis





# General Principles

## Newton mechanics

Conservation of momentum

Particle motion

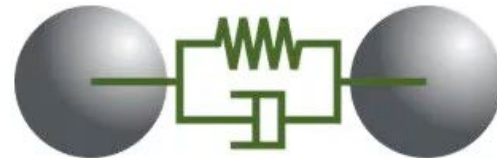
- **Translational**
- **Rotational**

Numerical integration with  
time steps

## Forces between particles

Force displacement laws

- Friction
- Stiffness





## Hard-sphere vs. Soft-sphere Methods

### Hard-sphere

Impulsive forces

Exchange of momentum

One collision at a time

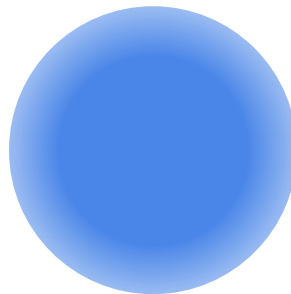


### Soft-sphere

Rigid particles but small overlaps allowed

Evaluates forces accurately

Simultaneous contacts possible





## Hard-sphere vs. Soft-sphere Methods

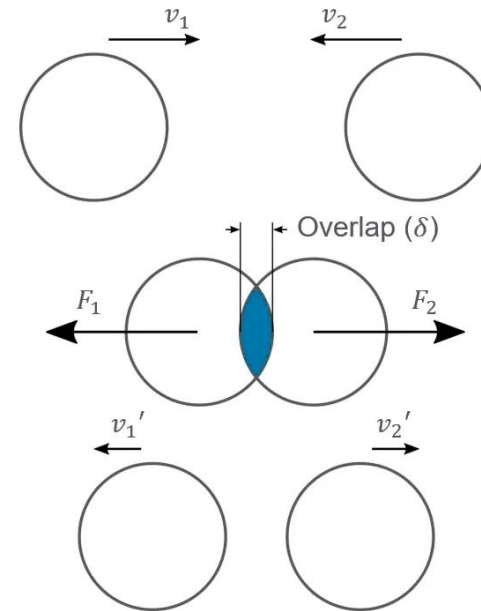
### Soft-sphere

Rigid particles but small overlaps allowed

Evaluates forces accurately

Simultaneous contacts possible

Most common  
and accurate





## Forces and Contact Force Models

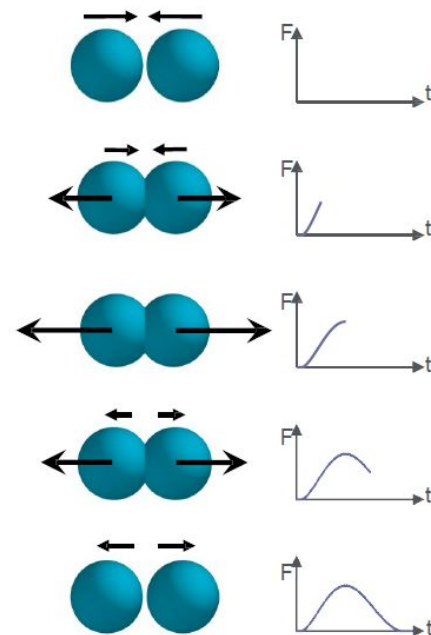
- **Contact force models**

The contact models relate the amount of overlap (tangential and normal) between two objects to determine the magnitudes of forces.

- **Other forces**

Forces resulting from particle collisions are not the only ones present in DEM.

The effects of particle body forces like gravity or noncontact forces like electrostatics or Van der Waals can also be simulated.





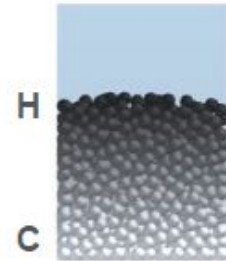
## Models for Various Material Behavior

- Different types of material behavior can be simulated by a range of well-established models with DEM.

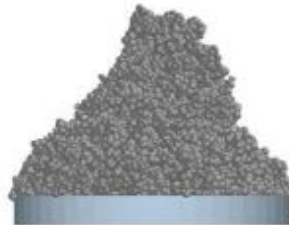
- Dry granular material



- Heat transfer



- Cohesion



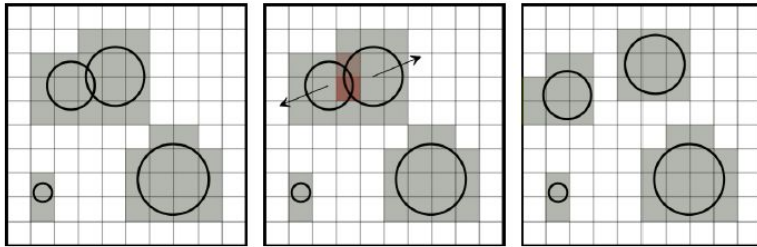
- Electrostatics



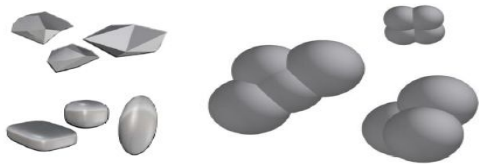


# Contact Detection Algorithms

- Importance
- Steps of a typical contact detection algorithm



- Particle shape Representations
- Three main irregular particle shapes used in DEM simulations





# Governing Equations: Particle Motion

Translational motion  $m\mathbf{a} = \Sigma\mathbf{F}$

$$m \frac{dv}{dt} = F_g + F_c + F_{nc}$$

$m$ : mass of particle

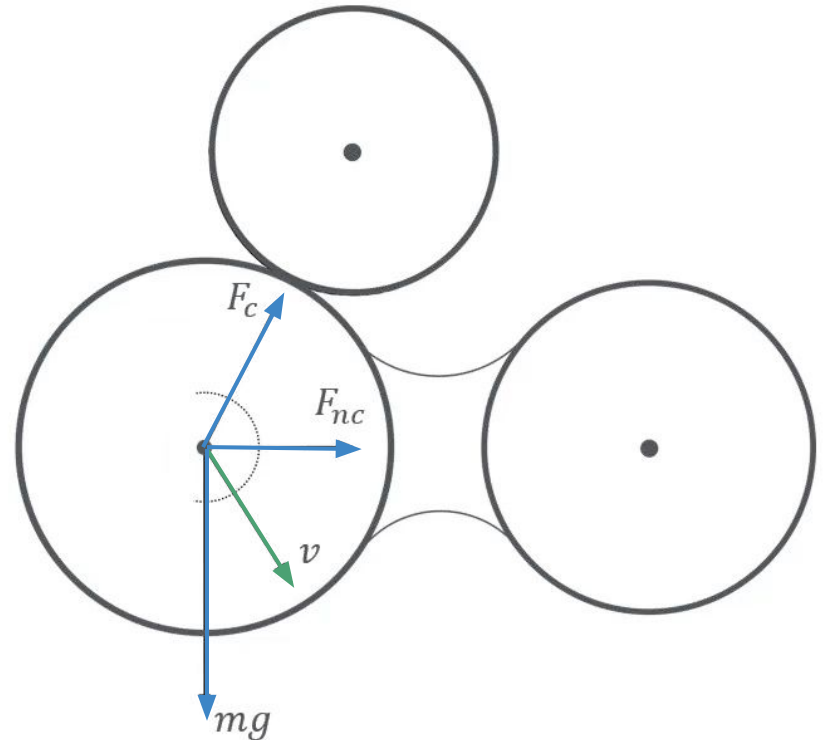
$a$ : translational acceleration

$v$ : translational velocity

$F_g$ : gravitational force

$F_c$ : contact forces between particles

$F_{nc}$ : non-contact forces





# Governing Equations: Particle Motion

## Rotational motion

$$I\alpha = M$$

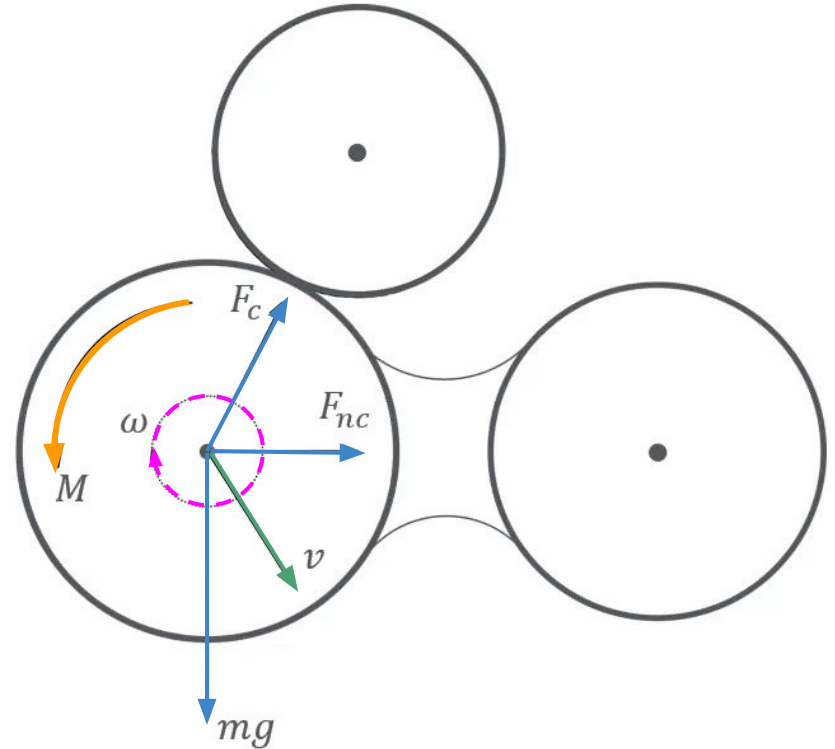
$$I \frac{d\omega}{dt} = M$$

$I$ : moment of inertia

$\alpha$ : angular acceleration

$\omega$ : angular velocity

$M$ : torque acting on particle







# Governing Equations: Other Forces

## Contact/non-contact forces to consider

- Friction between particles
- Contact plasticity (recoil)
- Gravity between particles
- Cohesion/adhesion
- Molecular forces
  - e.g. electrostatic, Coulomb

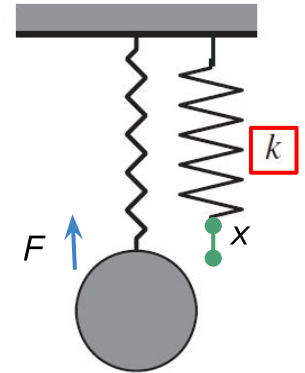
## Hooke's Law

$$F = -kx$$

F: restoring force

k: spring constant

x: extension length





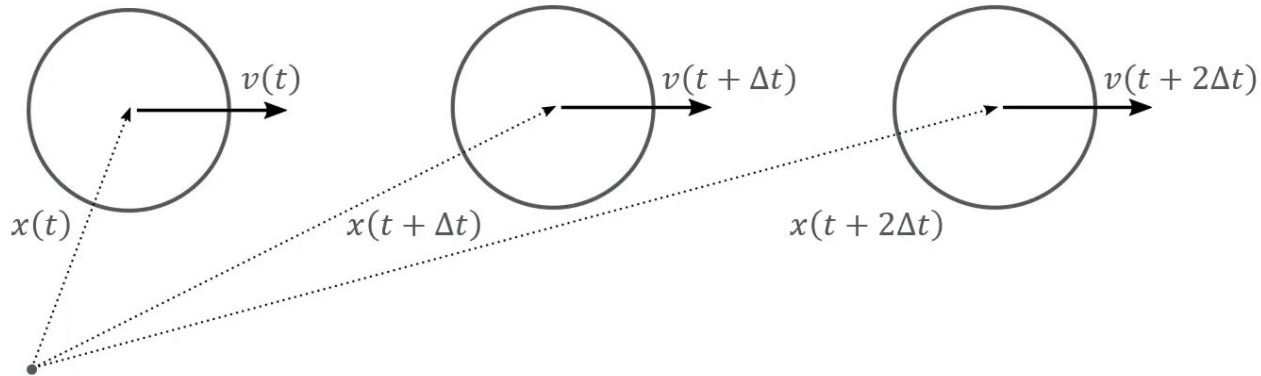
# Governing Equations: Time-stepping

## Numerical Integration

After computing accelerations, integrate over **time step ( $\Delta t$ )** to calculate particle **velocities** and **positions** (update at every time step)

$$\mathbf{x}(t + \Delta t) = \mathbf{x}(t) + \mathbf{v}(t)\Delta t$$

$$\mathbf{v}(t + \Delta t) = \mathbf{v}(t) + \mathbf{a}(t)\Delta t$$





# Importance of Time Steps

## Time steps chosen

Time step ( $\Delta t$ ) has to be chosen sufficiently small (1e-4 to 1e-6 s) for two main reasons: prevent excessive overlaps which result in unrealistically high forces and avoid effects of disturbance waves (Rayleigh waves).

## Rayleigh surface waves

$$T_R = \frac{\pi R(\rho/G)^{1/2}}{0.1631\nu + 0.8766}$$

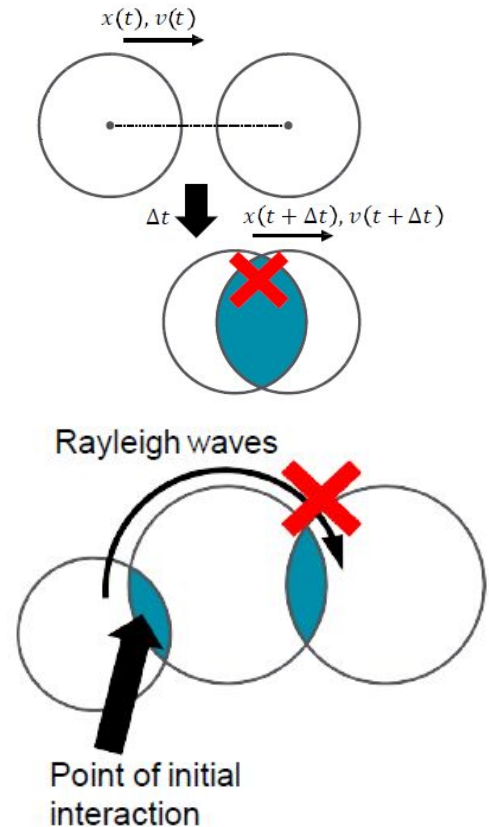
$T_R$  is the Rayleigh time step

$R$  is the particle radius

$\rho$  is the density

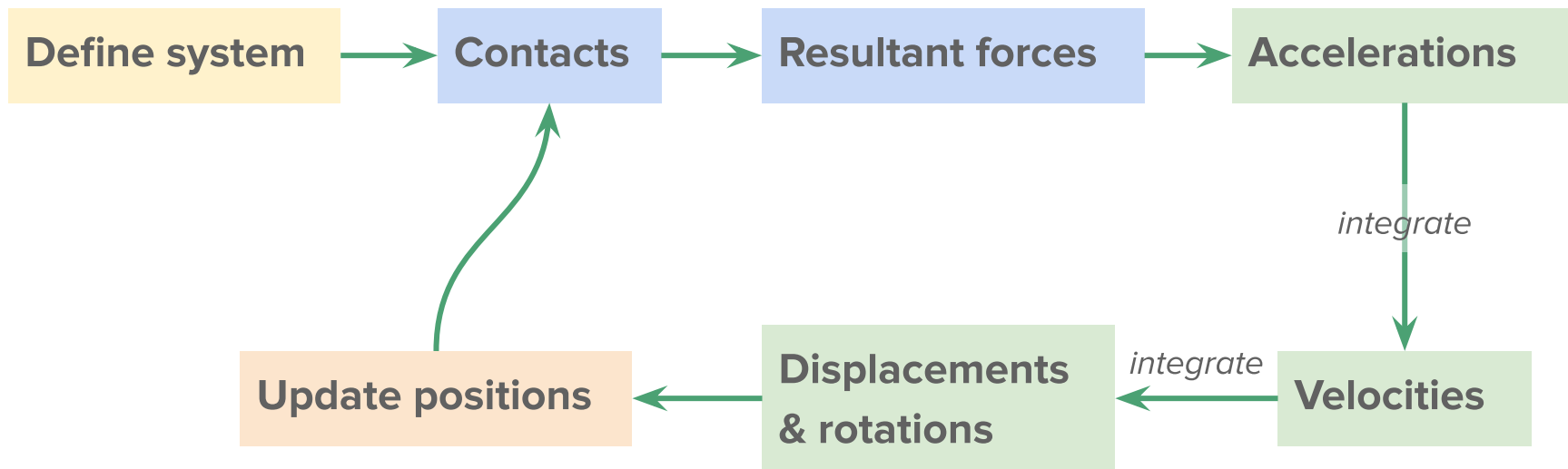
$G$  is the shear modulus

$\nu$  is the Poisson's ratio of the particle





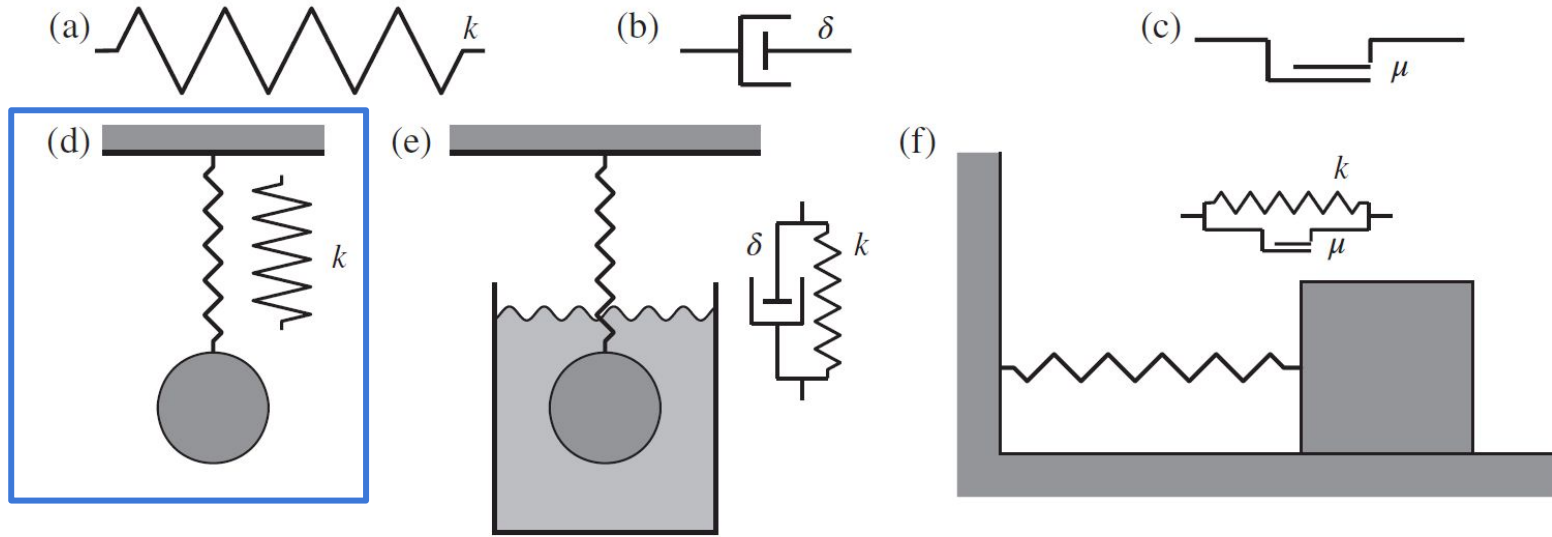
# Calculation Cycle





# Hand-Calculation Example

Six classic mechanical model used in DEM analysis



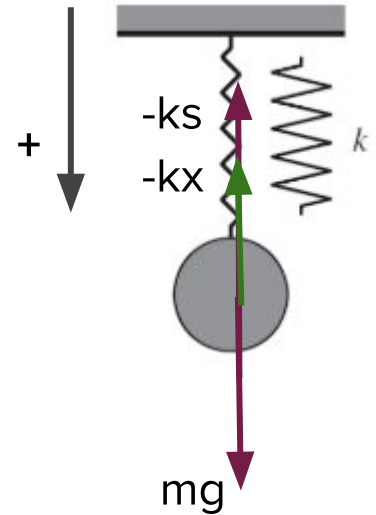
**Undamped linear oscillator and Damped linear oscillator:**



# Hand Calculation Example

## Undamped linear oscillator:

- At Equilibrium Point
  - a) Force created by Gravity  $F = mg$
  - b) Restoration force of the spring to oppose the pulling force by gravity  $F = -ks$
  - c) Restoration force of the spring  $F = -kx$
- The object release from the height above the equilibrium position at the beginning.





# Hand-Calculation Example

## Undamped linear oscillator:

Newton's Law: Total force applied to the body = Motion of the body

$$-kx + \underline{mg - ks} = 0$$

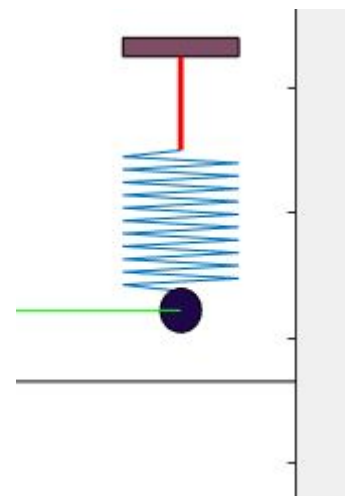
$$-kx$$

$$F = ma$$

$$a = \frac{d^2x}{dt^2}$$

$$ma = m \frac{d^2x}{dt^2}$$

$$-kx = m \frac{d^2x}{dt^2}$$





# Hand-Calculation Example

Undamped linear oscillator:

$$m \frac{d^2x}{dt^2} + kx = 0 \rightarrow x = e^{nt} \rightarrow mn^2 + k = 0 \rightarrow n_{1,2} = \pm \sqrt{-\frac{k}{m}} = \pm \sqrt{\frac{k}{m}} i = \pm \omega_0 i$$

$$x(t) = C_1 \cos(\omega_0 t) + C_2 \sin(\omega_0 t) \quad x(t) = A \cos(\omega_0 t - \phi)$$

$$\text{B.C. 1 : } x(t=0) = x_0 \rightarrow A = x_0$$

$$\text{B.C. 2 : } x(t \rightarrow \infty) = 0 \rightarrow \phi = \tan^{-1}\left(\frac{A \sin \phi}{A \cos \phi}\right)$$

$A$ : the amplitude of the displacement

$\phi$ : the phase shift or phase angle of the displacement.



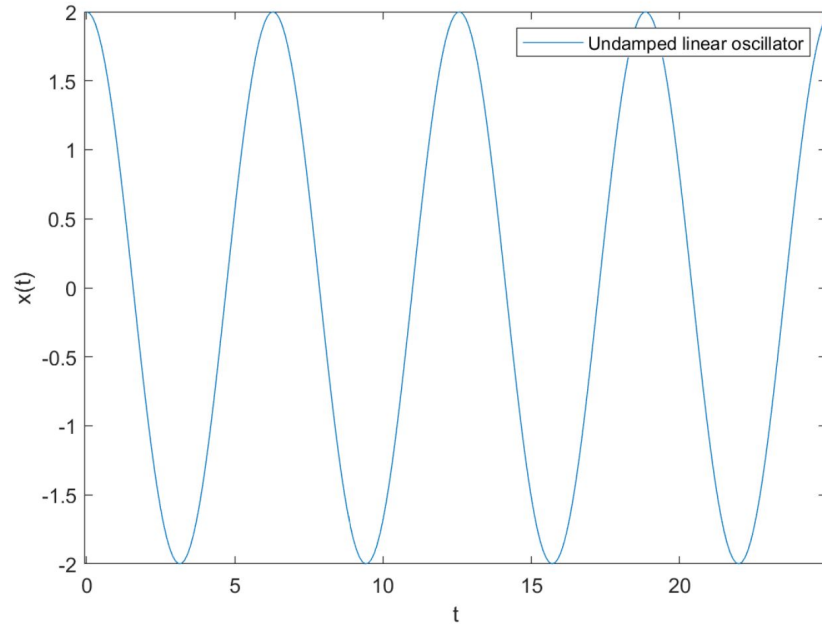


# Hand-Calculation Example

Undamped linear oscillator:

$$x(t) = A \cos\left(\sqrt{\frac{k}{m}}t - \phi\right)$$

$$v = \frac{dx}{dt} = -C \cdot \sin\left(\sqrt{\frac{k}{m}}t - \phi\right)$$





# Hand-Calculation Example

## Damped linear oscillator:

Newton's Law: Total force applied to the body = Motion of the body

$$F = ma$$

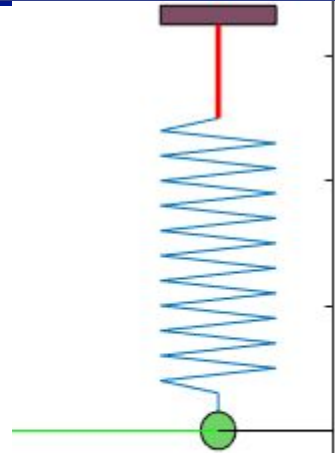
$$-kx + \underbrace{mg - ks}_{0} - \beta \frac{dx}{dt}$$

$$-kx - \beta \frac{dx}{dt}$$

$$a = \frac{d^2x}{dt^2}$$

$$ma = m \frac{d^2x}{dt^2}$$

$$-kx - \beta \frac{dx}{dt} = m \frac{d^2x}{dt^2}$$



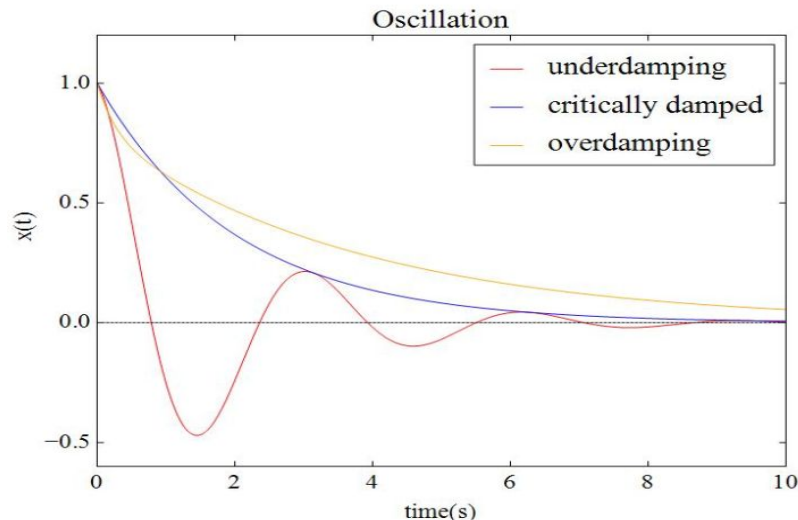


# Hand-Calculation Example

Damped linear oscillator:

$$m \frac{d^2x}{dt^2} + \beta \frac{dx}{dt} + kx \rightarrow x = e^{nt} \rightarrow mn^2 + \beta n + k = 0 \rightarrow n_{1,2} = -\frac{\beta \pm \sqrt{\beta^2 - 4mk}}{2m}$$

$$\begin{cases} \beta^2 - 4mk > 0, \text{Overdamped} \\ \beta^2 - 4mk = 0, \text{Critical damping} \\ \beta^2 - 4mk < 0, \text{Underdamped} \end{cases}$$





# Hand-Calculation Example

Damped linear oscillator:

$$n_{1,2} = -\frac{\beta \pm \sqrt{\beta^2 - 4mk}}{2m}$$

$$\omega = \frac{\sqrt{|\beta^2 - 4mk|}}{2m}$$

$$n_{1,2} = -\frac{\beta}{2m} \pm i\omega$$

$$x(t) = C_1 e^{n_1 t} + C_2 e^{n_2 t} = C_1 e^{\left(-\frac{\beta}{2m} + i\omega\right)t} + C_2 e^{\left(-\frac{\beta}{2m} - i\omega\right)t}$$

$$x(t) = C_1 e^{t\left(-\frac{\beta}{2m}\right)} * e^{i\omega t} + C_2 e^{t\left(-\frac{\beta}{2m}\right)} * e^{-i\omega t}$$

$$x(t) = e^{t\left(-\frac{\beta}{2m}\right)} * (C_1 e^{i\omega t} + C_2 e^{-i\omega t})$$

$$x(t) = e^{t\left(-\frac{\beta}{2m}\right)} * [C_1' \cos(\omega t) + C_2' \sin(\omega t)]$$

$$x(t) = A e^{t\left(-\frac{\beta}{2m}\right)} * \cos[\omega t - \phi]$$



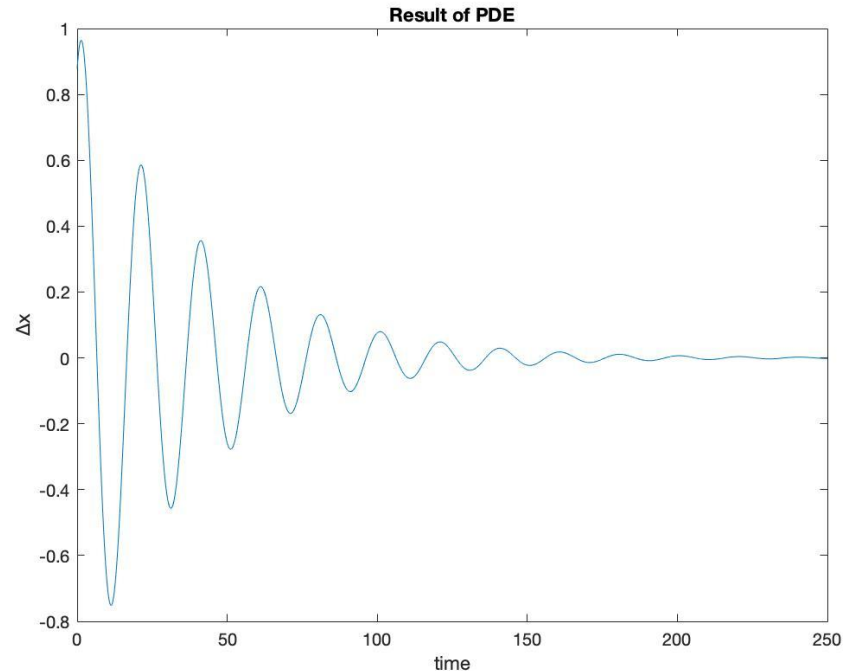
# Hand-Calculation Example

Damped linear oscillator:

$$x(t) = Ae^{t\left(-\frac{\beta}{2m}\right)} * \cos[\omega t - \phi]$$

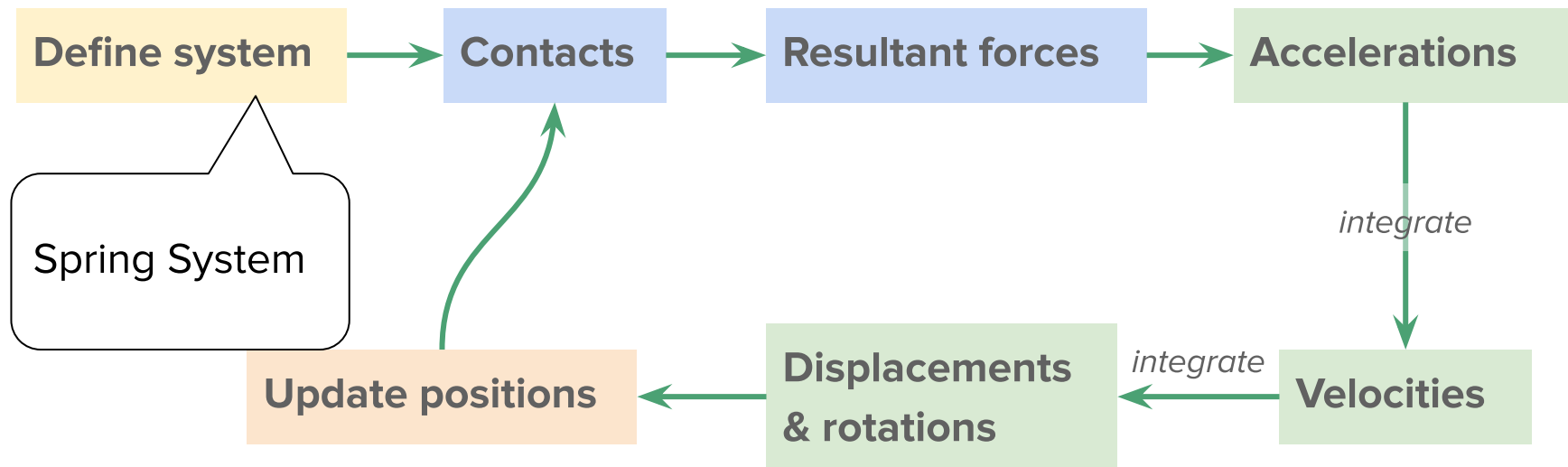
$A$ : the amplitude of the displacement

$\phi$ : the phase shift or phase  
angle of the displacement.



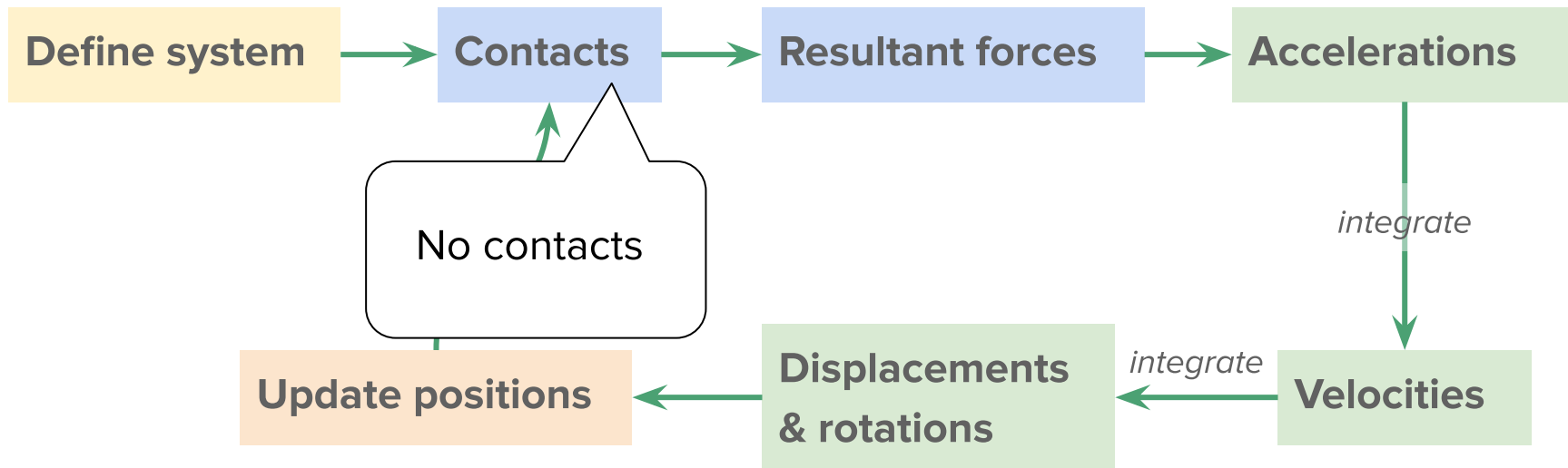


# Calculation Cycle



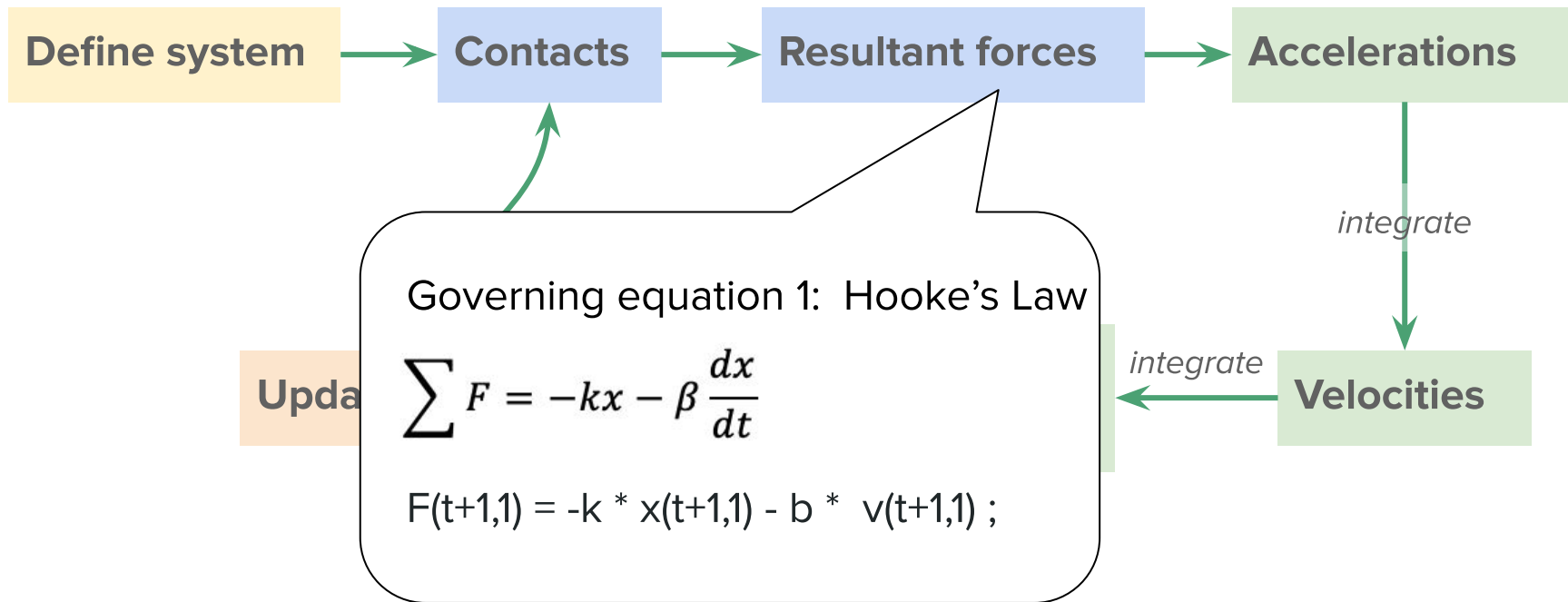


# Calculation Cycle





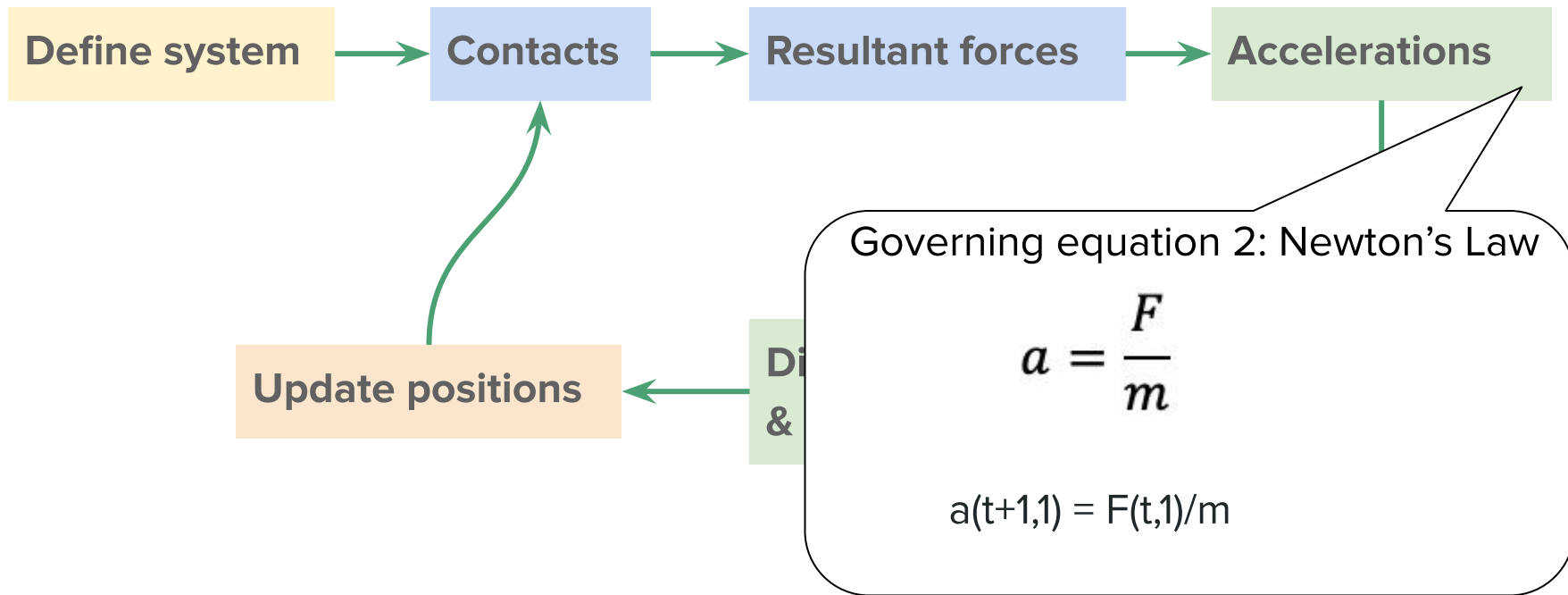
# Calculation Cycle





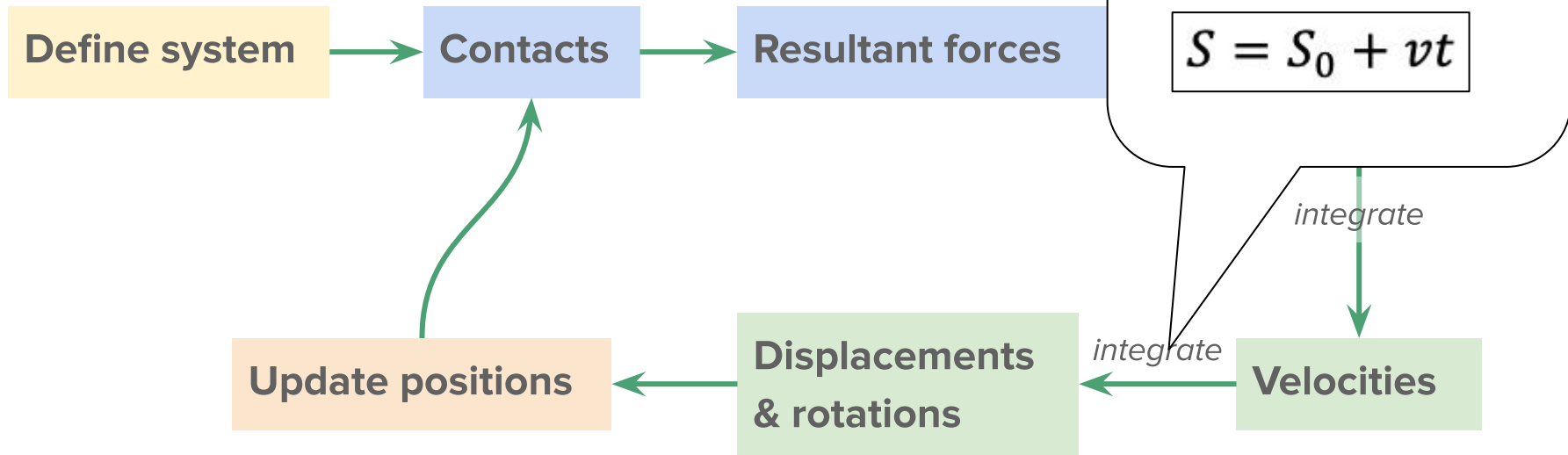


# Calculation Cycle





# Calculation Cycle





# Hand-Calculation Example

## Code

```
g = 9.81;
```

```
k = 0.2;
```

```
m = 2;
```

```
b = 0.3;
```

```
TimeStep = 0.1;
```

```
x = zeros(1000,1);
```

```
x(1,1) = -1;
```

```
x(2:1000,1) = nan;
```

```
v = zeros(1000,1);
```

```
v(1,1) = 0;
```

```
v(2:1000,1) = nan;
```

```
F = zeros(1000,1);
```

```
F(1,1) = (k * x(1,1)-b*v(1,1));
```

```
F(2:1000,1) = nan;
```

```
time = zeros(1,1000);
```

```
a = zeros(1000,1);
```



# Hand-Calculation Example

## Code

```
g = 9.81;  
k = 0.2;  
m = 2;  
b = 0.3;  
TimeStep = 0.1;
```

```
x = zeros(1000,1);  
x(1,1) = -1;  
x(2:1000,1) = nan;
```

```
v = zeros(1000,1);  
v(1,1) = 0;  
v(2:1000,1) = nan;
```

```
F = zeros(1000,1);  
F(1,1) = (k * x(1,1) - b * v(1,1));  
F(2:1000,1) = nan;
```

```
time = zeros(1,1000);  
a = zeros(1000,1);
```



# Hand-Calculation Example

## Code

```
for t = 1:1:1000
    time(1,t+1) = t * TimeStep ;

    a(t+1,1) = F(t,1)/m;

    v(t+1,1) = v(t,1) + a(t,1) * TimeStep;

    x(t+1,1) = x(t,1) + v(t,1)*TimeStep;

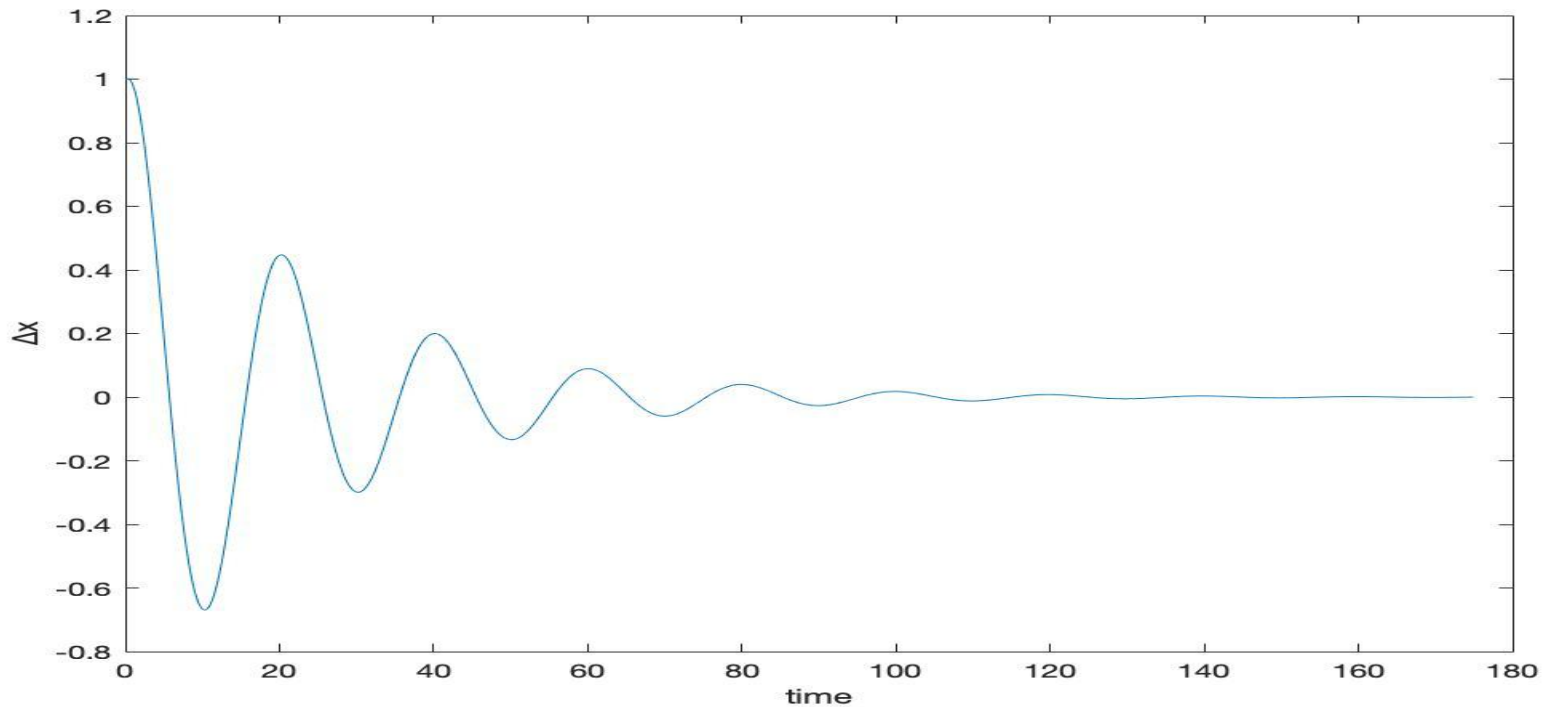
    if x(t+1,1) > 0
        if x(t+1,1) <= 15
            F(t+1,1) = -k * x(t+1,1) - b * v(t+1,1) ;
        else
            x(t+1,1) = 15;
            v(t+1,1) = 0;
            F(t+1,1) = -k * x(t+1,1);
        end
    elseif x(t+1,1) >= -15
        F(t+1,1) = (-k * x(t+1,1) - b * v(t+1,1));
    else
        x(t+1,1) = -15;
        v(t+1,1) = 0;
        F(t+1,1) = -k * x(t+1,1);
    end

end

plot(time,-x);
```



# Hand-Calculation Example





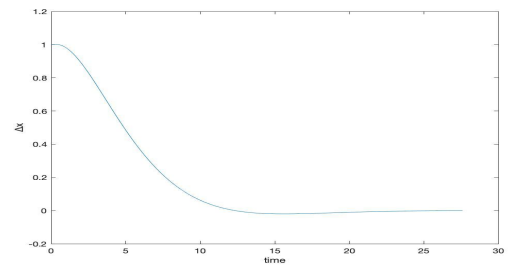
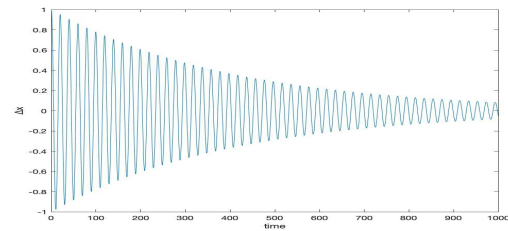
# Hand-Calculation Example $k = 0.2$

Damping Coefficient	Time	Figure
0.2	174.7	
0.4	78	
0.6	40.1	
0.8	33.6	



# Hand-Calculation Example $k = 0.2$

Damping Coefficient	Time	Figure
0.05	>1000	
0.99	27.6	







# Hand-Calculation Example $b = 0.2$

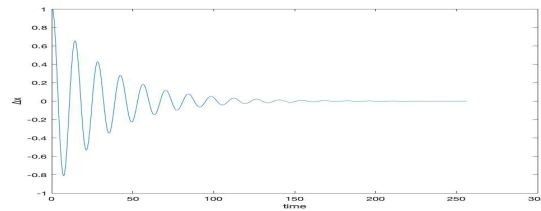
Spring constant

Time

Figure

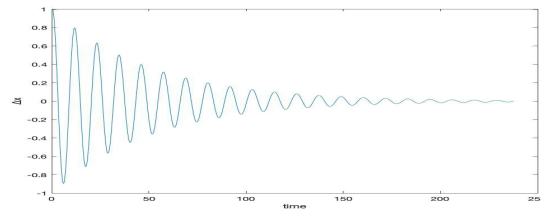
0.4

212



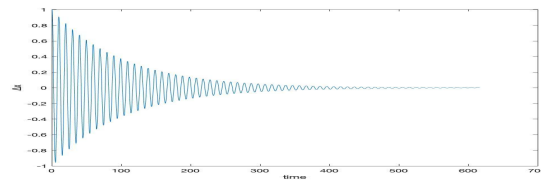
0.6

237.5



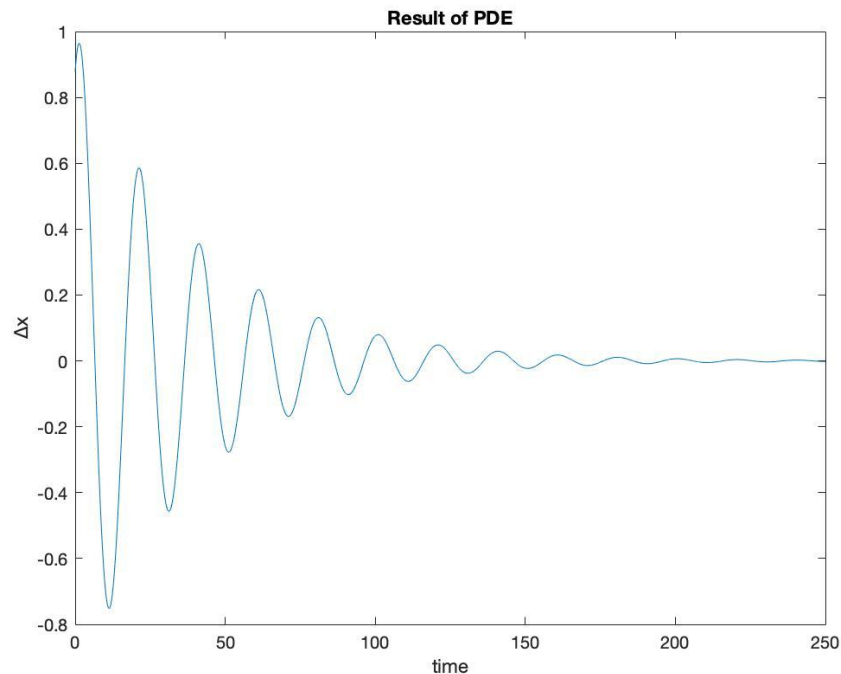
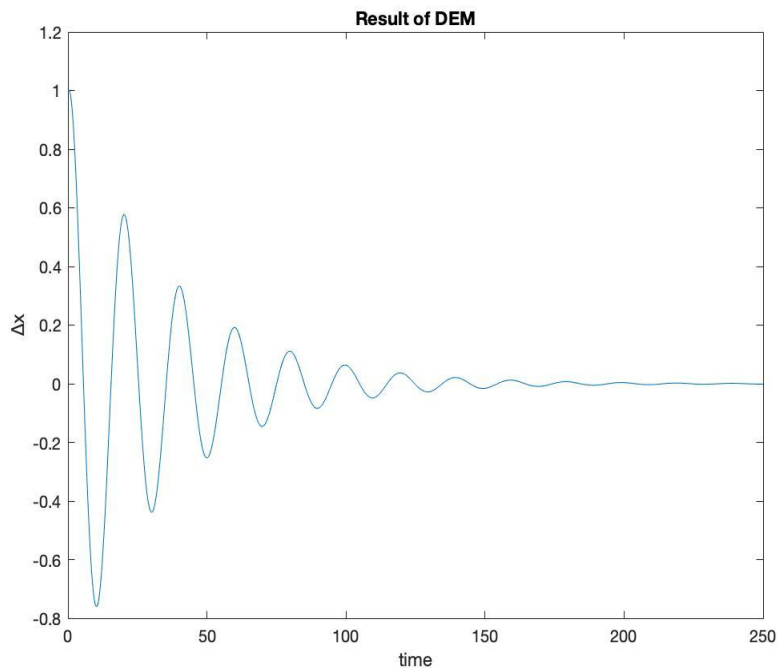
0.8

616.6





# Comparison between DEM result and PDE result





Basic assumption:

- ❖ Particles are soft
- ❖ Continuity of force
- ❖ Viscous damping proportional to velocity

**Damped/Dissipated**

$$F(x) = \begin{cases} mg - Dv & \text{for } x \geq 0 \\ mg - kx - Dv & \text{for } x < 0 \end{cases}$$

**Govern Equation**

**Without Dissipation**

$$F(x) = \begin{cases} mg & \text{for } x \geq 0 \\ mg - kx & \text{for } x < 0 \end{cases}$$

```
function [dy]=bouncing_ball(t,y)
% bouncing ball without dissipation
global g
global k
if (y(2)>=0)
dy=[g
y(1)];
else
dy=[g-k*y(2)
y(1)];
end
return
```

Parameters:

K is set to be 100 and D is set to be 1.

The initial height is set to be 4

The time range is set to be [0 10]



## Bouncing ball without dissipation

```
clear, format compact
global k, k=100;
global g, g=-9.81;
global D, D=1;
x0=4
v0=1
```

```
tspan=[0 10]
```

```
[t,y]=ode23('bouncing_ball',tspan,[v0 x0]);
[t1,y1]=ode23('bouncing_balldis',tspan,[v0 x0]);
```

```
figure(1)
plot(t,y(:,2),'ro', t1,y1(:,2), 'bo')
legend('Bouncing ball without dissipation', ...
       'Bouncing ball with dissipation')
```

```
function [dy]=bouncing_ball(t,y)
% bouncing ball without dissipation
global g
global k
if (y(2)>=0)
dy=[g
y(1)];
else
dy=[g-k*y(2)
y(1)];
end
return
```

```
function varargout = ode23(ode,tspan,y0,options,varargin)
%ODE23 Solve non-stiff differential equations, low order method.
```



## Bouncing ball with dissipation

```
clear, format compact
global k, k=100;
global g, g=-9.81;
global D, D=1;
x0=4
v0=1

tspan=[0 10]
% [t,y]=ode23('bouncing_ball',tspan,[v0 x0]);
[t1,y1]=ode23('bouncing_balldis',tspan,[v0 x0]);

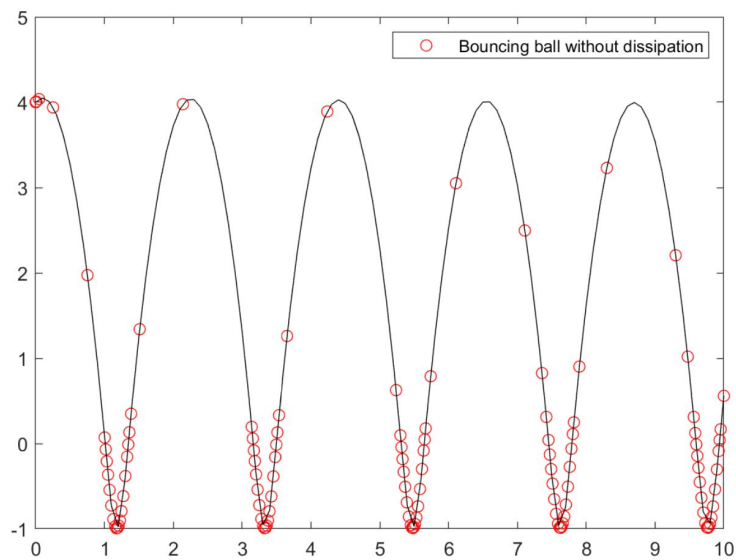
figure(1)
plot(t,y(:,2),'ro', t1,y1(:,2), 'bo')
legend('Bouncing ball without dissipation', ...
       'Bouncing ball with dissipation')
```

```
function [dy]=bouncing_balldis(t,y)
% bouncing ball without dissipation
global g
global k
global D
f_el=-k*y(2)
f_damp=-D*y(1)
f_tot=f_el+f_damp
if (sign(f_tot*f_el)<0)
f_tot=0
end

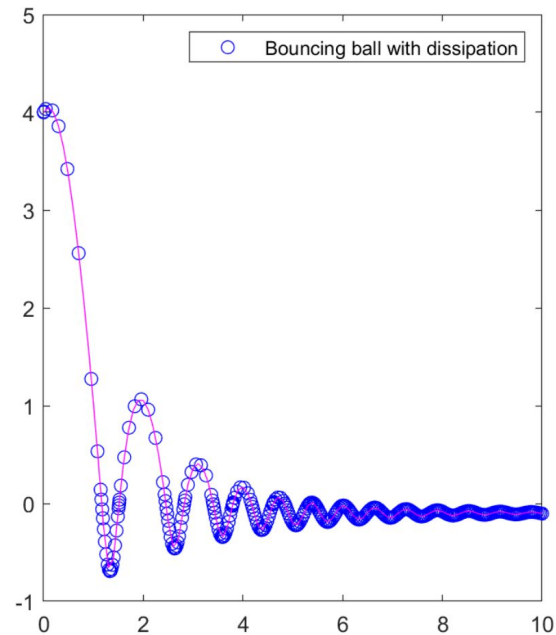
if (y(2)>=0)
dy=[g-D*y(1)
y(1)];
else
dy=[g+f_tot
y(1)];
end
return
```



### Bouncing ball without dissipation

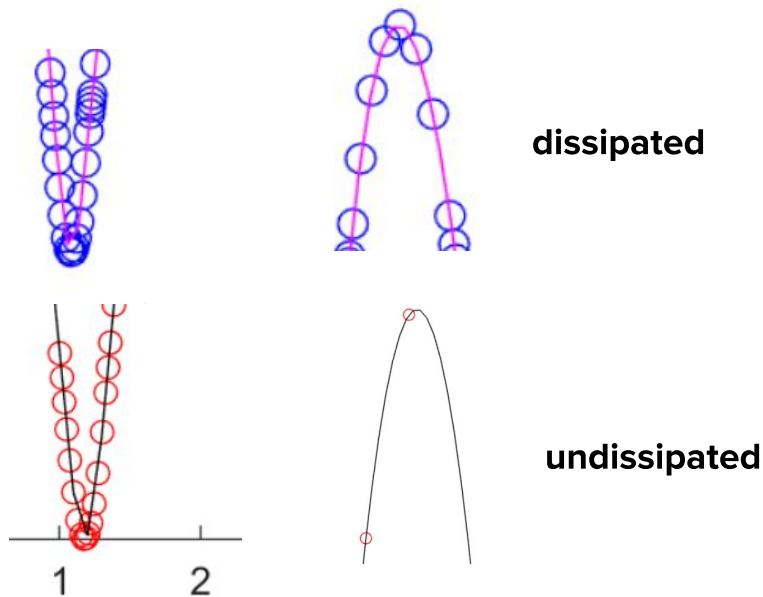
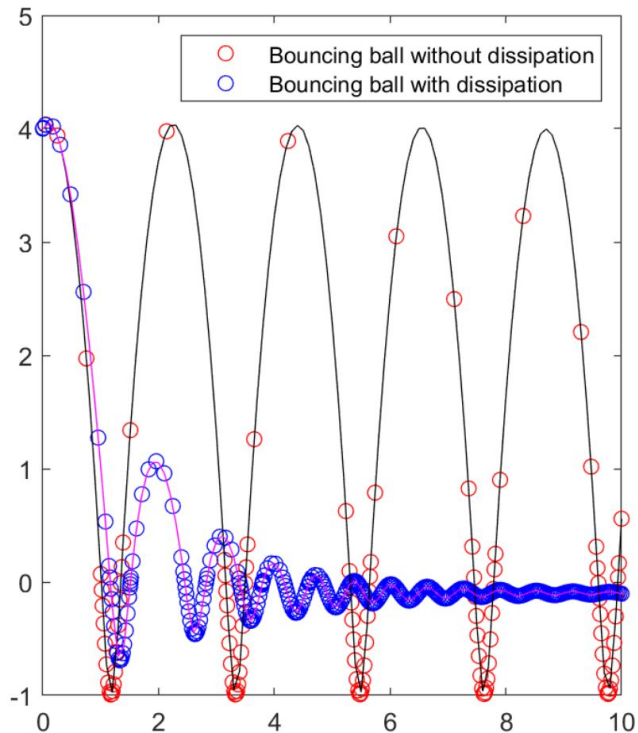


### Bouncing ball with dissipation





## Comparison



➤ Bouncing ball with dissipation have a smaller time steps



### Basic assumption:

- ❖ No dissipation /No damping
- ❖ Soft and round particle
- ❖ Perfect Elastic collision
- ❖ Rotation is ignored

### External Force:

- ❖ Gravity force
- ❖ Elastic force

### Initialization:

- ❖ Particle number (1 to 5)
- ❖ Same radius(0.5) , mass(1)
- ❖ Same Young Modulus (1000)
- ❖ Minimum(0) and Maximum vertical displacement( $2*n+2$ )
- ❖ Initial vertical position and velocity:  
 $y_0:[2 \ 0 \ 4 \ 0 \ \dots \ 10 \ 0]$
- ❖ End time: 4





# Numerical Example

## Modeling of Polygonal Particle - 1D

```
clear all
format compact
n_part=2
% initialize radius and mass
global rad, rad(1:n_part)=0.5;
global m, m(1:n_part)=1;
global E, E=1000; % Young's modulus
global lmax, lmax=2*n_part+2;
global lmin, lmin=0;
global g, g=-9.81;

% initialize positions and velocities=0
r0=2*[1:n_part];
v0=r0*0;
y0(1:2:2*n_part-1)=r0;
y0(2:2:2*n_part)=v0;
t_end=4

[t,y]=ode113('DEMround1D',[0 t_end],y0);
hold on
for i=1:n_part
plot(t,y(:,2*i-1),'bo-')
end
axis([0 max(t) lmin-.5 lmax+.5])
```

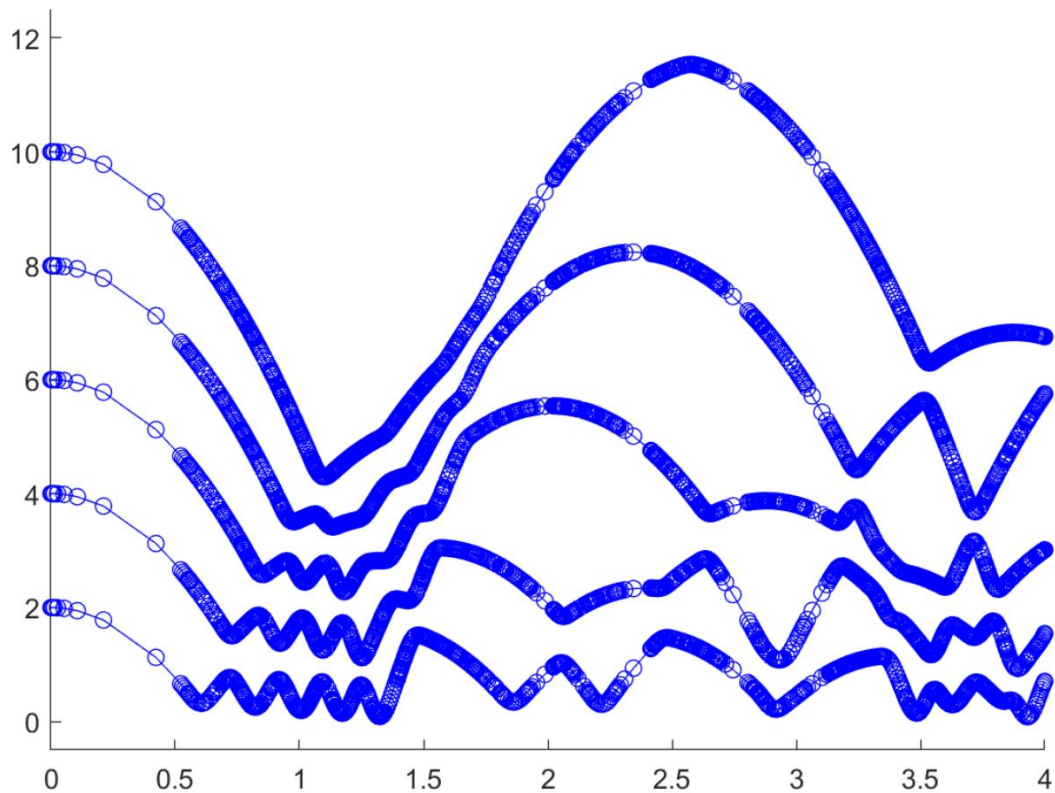
```
function [dydt]=DEMround1D(t,y);
global m rad E lmax lmin g
n_part=length(m);
if length(y)~=2*length(m)
error('length of y must be twice the length of m')
end
if length(rad)~=length(m)
error('length of r must be twice the length of m')
end
a=zeros(1,n_part);
for i_part=1:n_part
x1=y(2*i_part-1); % position of first particle
rad1=rad(i_part);
% Particle-Particle Interaction
for j_part=i_part+1:n_part
x2=y(2*j_part-1); % position of second particle
rad2=rad(j_part);
if (abs(x2-x1)<(rad(i_part)+rad(j_part))) % overlap
forcemagnitude=E*abs(abs(x1-x2)-(rad1+rad2));
forcedirection=sign(x1-x2);
f=forcemagnitude*forcedirection;
a(i_part)=a(i_part)+f;
a(j_part)=a(j_part)-f; % use action=reaction
end
end
% Particle-wall Interaction
if (x1-rad1)<lmin
a(i_part)=a(i_part)-E*((x1-rad1)-lmin);
end
```

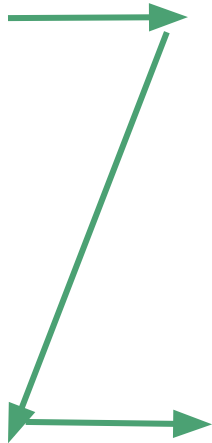
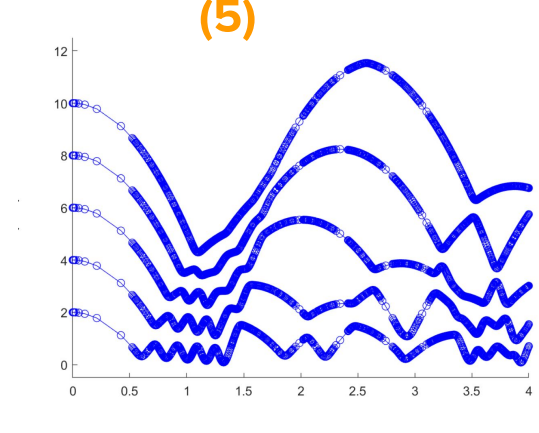
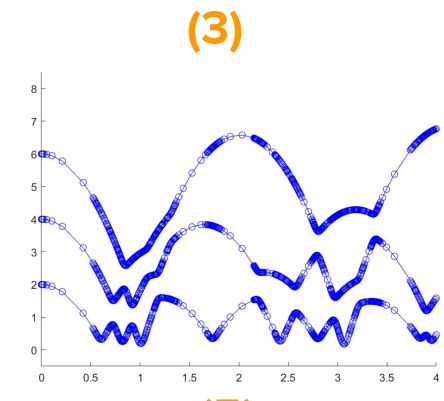
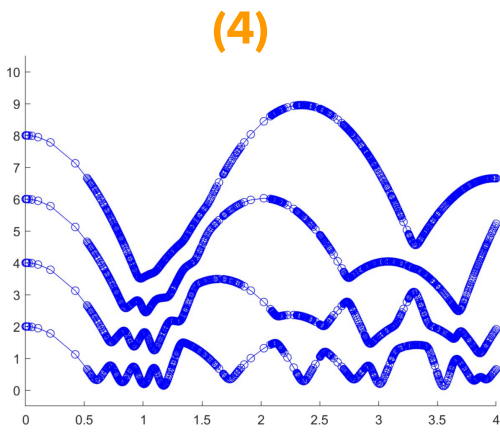
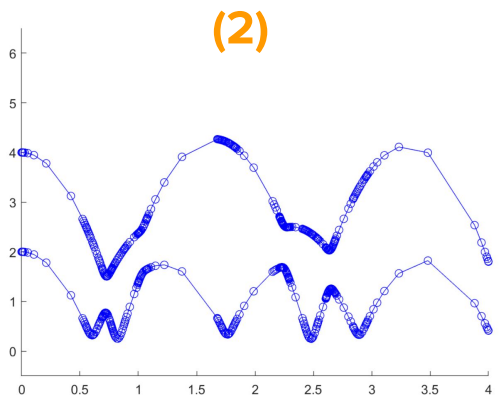
particle interaction

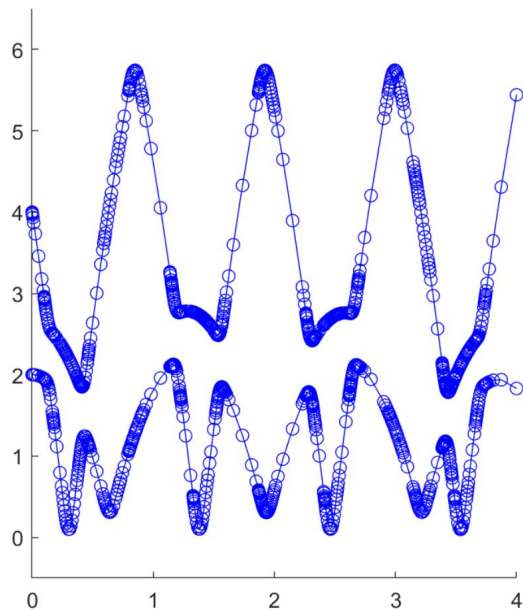
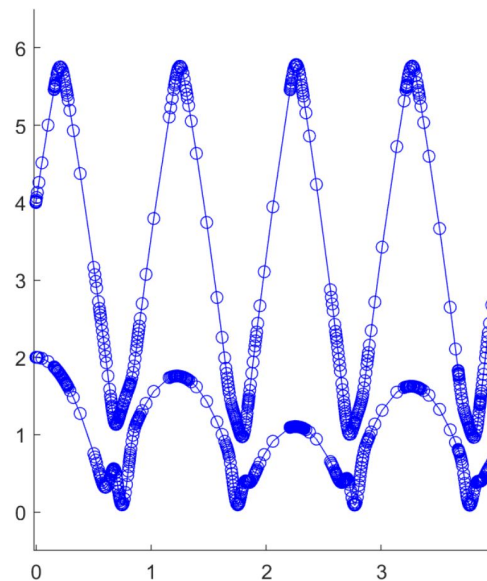




## Result

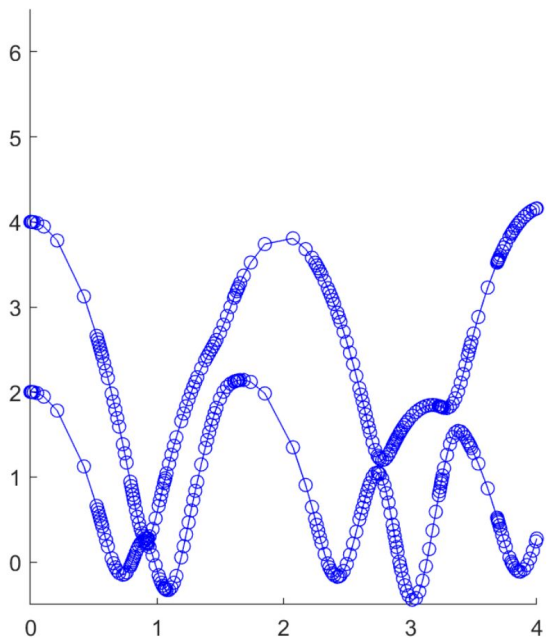




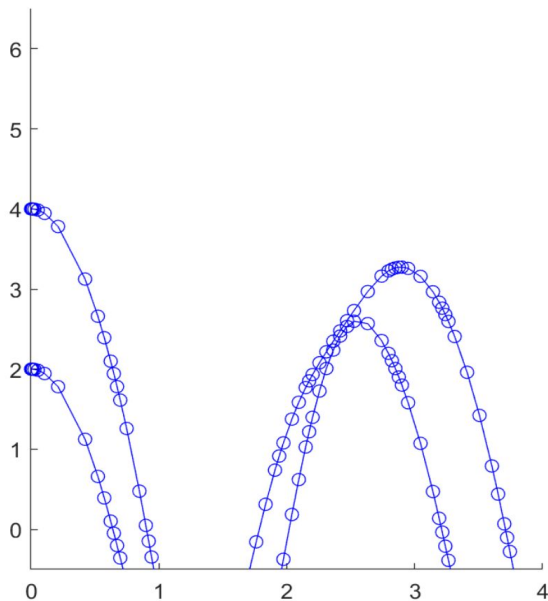
 $Y_0: [2 \ 0 \ 4 \ -10]$  $Y_0: [2 \ 0 \ 4 \ 10]$ 



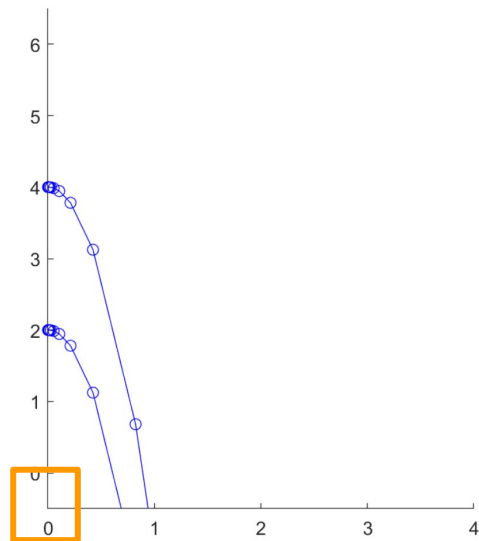
$E = 100$



$E = 10$



$E = 0$



**Initialization:**

- ❖ Particle number (18)
- ❖ Same radius(0.5) , mass(1)
- ❖ Same Young Modulus (10000)
- ❖ Minimum(0) and Maximum vertical displacement( $2*n+2$ ). Maximum and minimum horizontal displacement is the same with vertical.
- ❖ Initial position and velocity vector  
y0:[X1 randx1 Y1 randy1 ... X18 randx18 Y18 randy18]:  $72*1 = 18 * 4$
- ❖ End time: 5





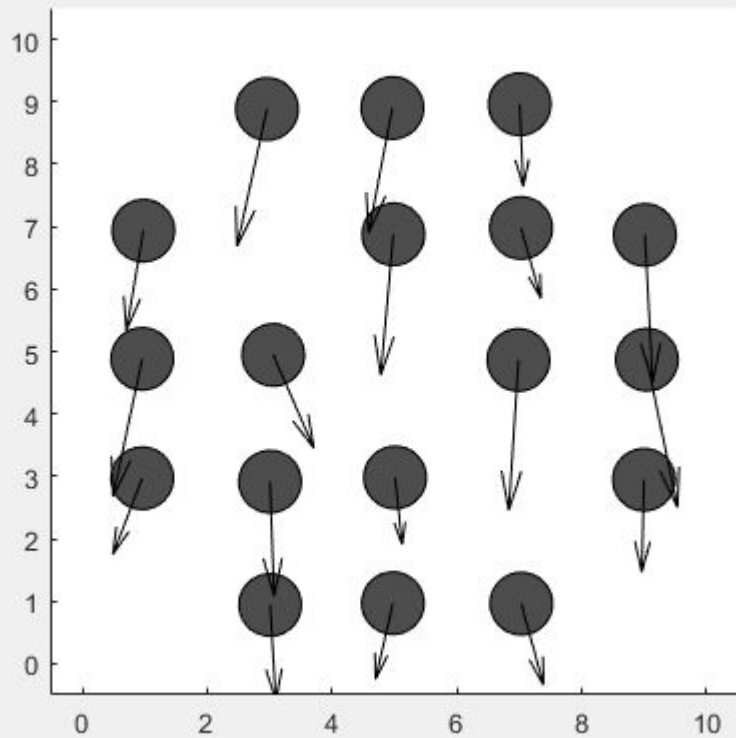
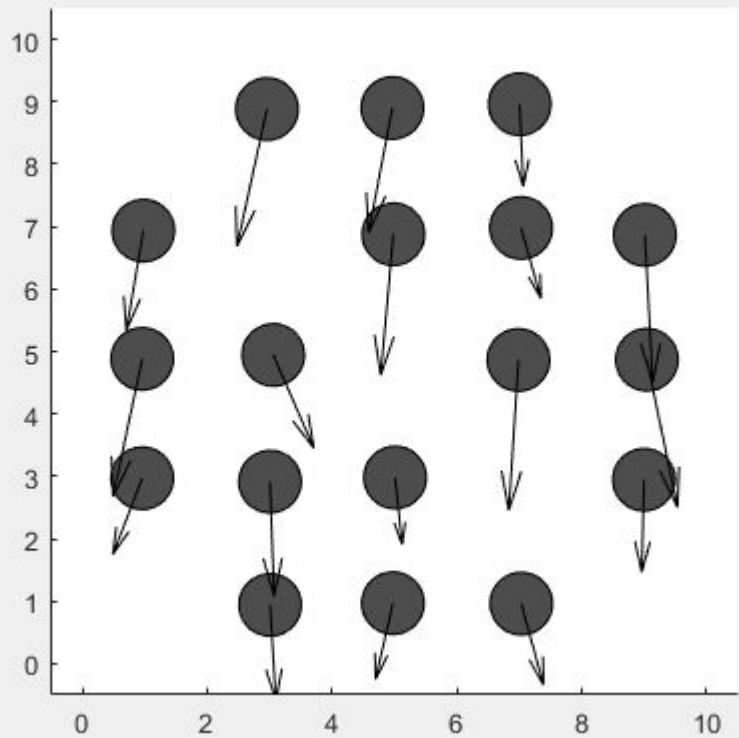
```
clear all
format compact
clf
n_part=18
% initialize radius and mass
global rad, rad(1:n_part)=0.5;
global m, m(1:n_part)=1;
global E, E=10000; % Young's modulus
global lmaxx, lmaxx=n_part/2+1;
global lminx, lminx=0;
global lmaxy, lmaxy=n_part/2+1;
global lminy, lminy=0;
global g, g=-9.81;
% initialize positions and velocities=0
rand('seed',5)
r0_x=2*mod([1:n_part],5)+1;
v0_x=rand(size(r0_x))-0.5;
r0_y=sort(r0_x);
v0_y=rand(size(r0_y))-0.5;
y0(1:4:4*n_part-3)=r0_x;
y0(2:4:4*n_part-2)=v0_x;
y0(3:4:4*n_part-1)=r0_y;
y0(4:4:4*n_part)=v0_y;
t_end=5;
[t,y]=ode113('DEMround2Dnorot',[0:0.005:t_end],y0);
figure(1)
[X,Y,Z]=cylinder(rad,55); % outline for plotting
```

```
function [dydt]=DEMround2Dnorot(t,y);
global m rad E lmax lmin lmaxx lminx lmaxy lminy g
n_part=length(m);
if length(y)~=4*length(m)
error('length of y must be four times the length of m')
end
if length(rad)~=length(m)
error('length of r must be four times the length of m')
end
a=zeros(2,n_part);
for i_part=1:n_part
r1=[y(4*i_part-3)
y(4*i_part-1)]; % position of first particle
rad1=rad(i_part);
% Particle-Particle Interaction
for j_part=i_part+1:n_part
r2=[y(4*j_part-3)
y(4*j_part-1)]; % position of second particle
rad2=rad(j_part);
if (norm(r1-r2)<(rad(i_part)+rad(j_part)))
forcemagnitude=E*abs(norm(r1-r2)-(rad1+rad2));
forcedirection=(r1-r2)/norm(r1-r2);
f=forcemagnitude*forcedirection;
a(:,i_part)=a(:,i_part)+f;
a(:,j_part)=a(:,j_part)-f;
end
end
% Particle-wall Interaction
if (r1(1)-rad1)<lminx
a(1,i_part)=a(1,i_part)-E*((r1(1)-rad1)-lminx);
```



# Numerical Example

Modeling of Polygonal Particle - 2D







# Application: Proppant Deformation

- Hydraulic Fracture
- Proppant Deformation
  - solid material (sand, ceramic), keep fracture open
- LIGGGHTS (open source)
- assumptions
  - shape is spherical
  - each grain is individual
  - can affect each other only touching points



## Types of Proppant



Ottawa Frac Sand



LiteProp™ 108 ULWP



Low Density Ceramic



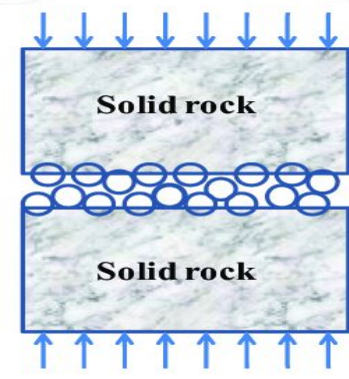
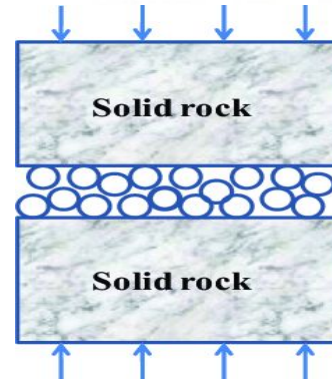
Brown Frac Sand



Resin-Coated Sand

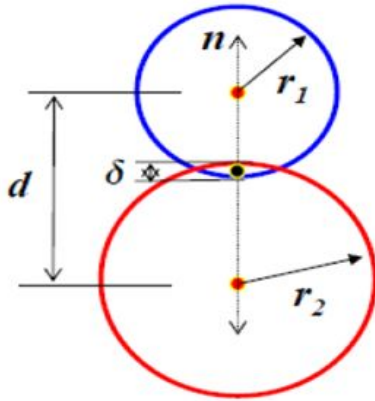


Sintered Bauxite





# Application

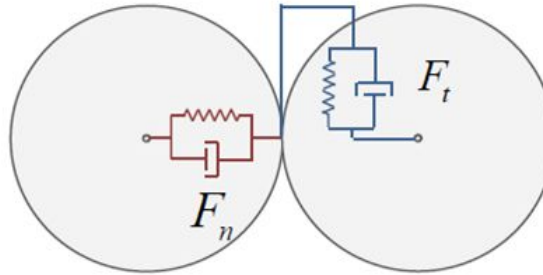


Overlap between two particles

$$\delta = r_1 + r_2 - d$$

- The magnitude of tangential force is bounded by the friction force

$$\max(|\vec{F}_t|) = |\mu \vec{F}_n|$$



Linear spring dashpot model assumed for DEM

$$\vec{F}_n = k_n \vec{\delta} + c_n \overline{\Delta v_n}$$

$$\vec{F}_t = k_t \left| \int_{t_{c,0}}^t \Delta v_t(\tau) d\tau \right| \vec{t} + c_t \overline{\Delta v_t}$$

Where:

$\Delta v_n$  = normal relative velocity at the contact point

$\Delta v_t$  = relative tangential velocity

$\vec{t}$  = tangential vector at the contact point

$k_n$  = elastic contact for normal contact

$k_t$  = elastic contact for tangential contact

$t_{c,0}$  = time at which the contact begins

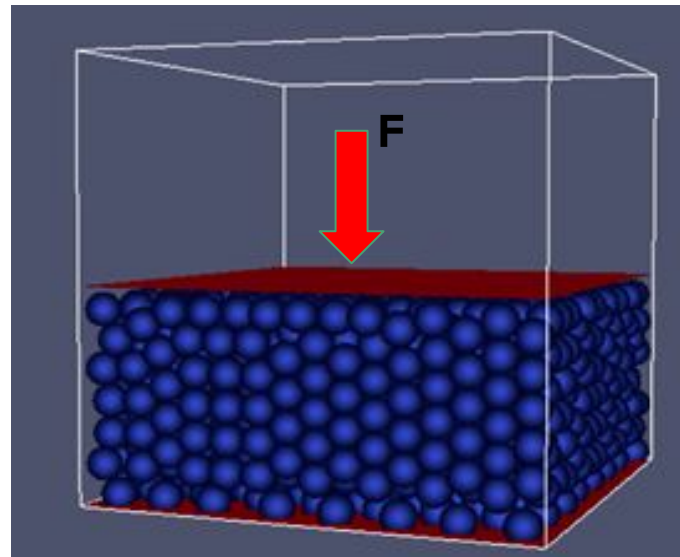


## Simulation

- 3x3x3 mm cubic shape
- proppant types and their features

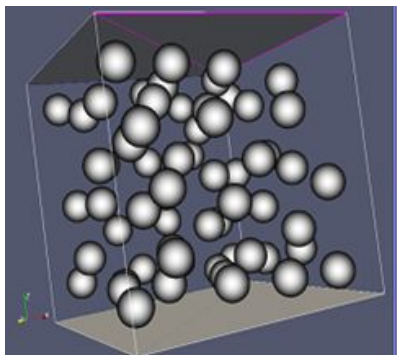
Proppant Type	Poisson's Ratio	Young's Modulus (Gpa)
Silica Sand	0.17	70
Resin Coated	0.19	220
Ceramic	0.22	375

- three diameter (0.005 mm, 0.01 mm, 0.015 mm)
- grains are elastic
- no temperature affect





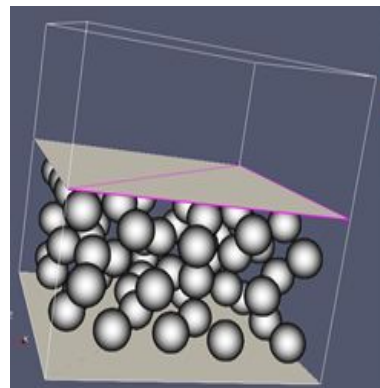
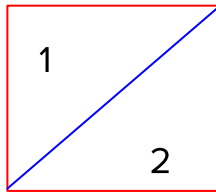
# Application (Simulation)



Cell ID	Cell Type	normal_stress_average	shear_stress_average
0,0	Triangle	0	0
1,1	Triangle	0	0
2,2	Triangle	0	0
3,3	Triangle	0	0

Activate Window

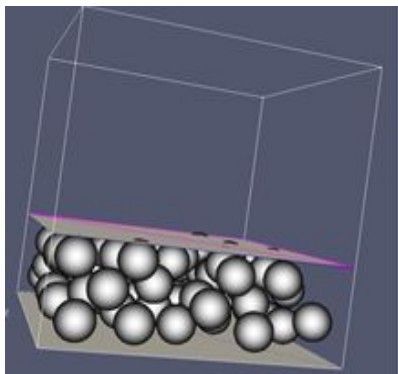
Step-1



Cell ID	Cell Type	normal_stress_average	shear_stress_average
0,0	Triangle	5108.75	648.49
1,1	Triangle	5870.79	479.03
2,2	Triangle	7585.89	371.245
3,3	Triangle	11368.8	1744.83

Activate Windows

Step -2



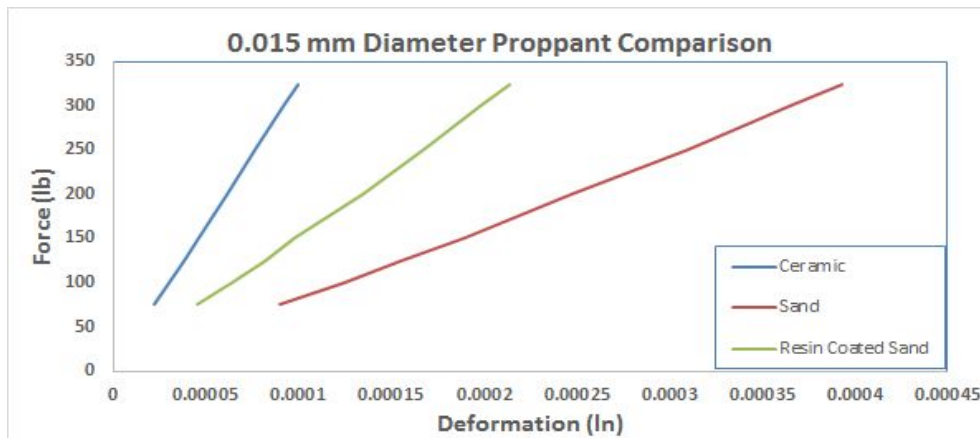
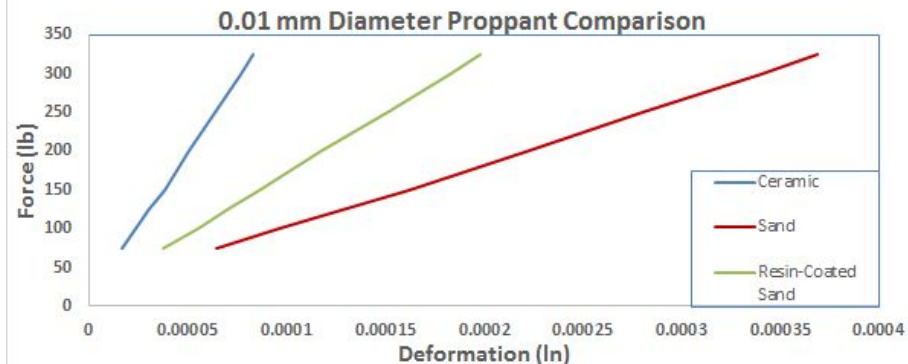
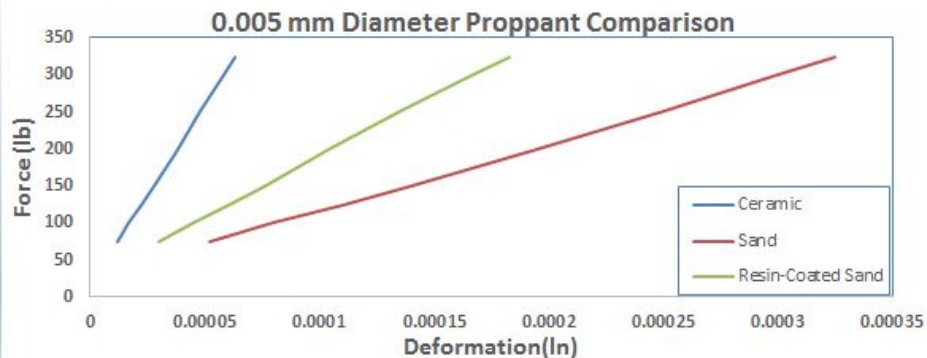
Cell ID	Cell Type	normal_stress_average	shear_stress_average
0,0	Triangle	15795.3	32862.4
1,1	Triangle	27300.5	9635.49
2,2	Triangle	30207.3	7801.48
3,3	Triangle	40230.8	52208.9

Activate Windows

Step-3

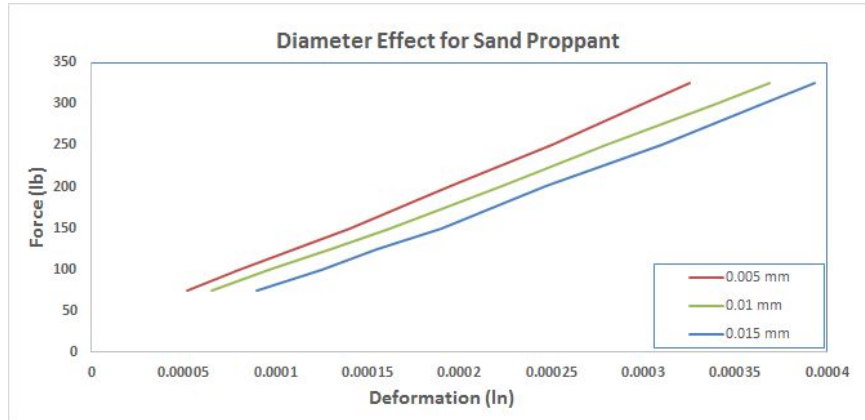
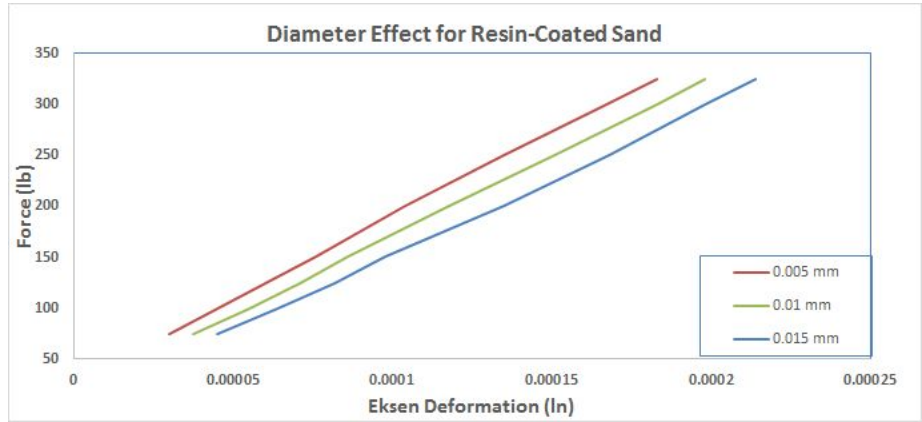
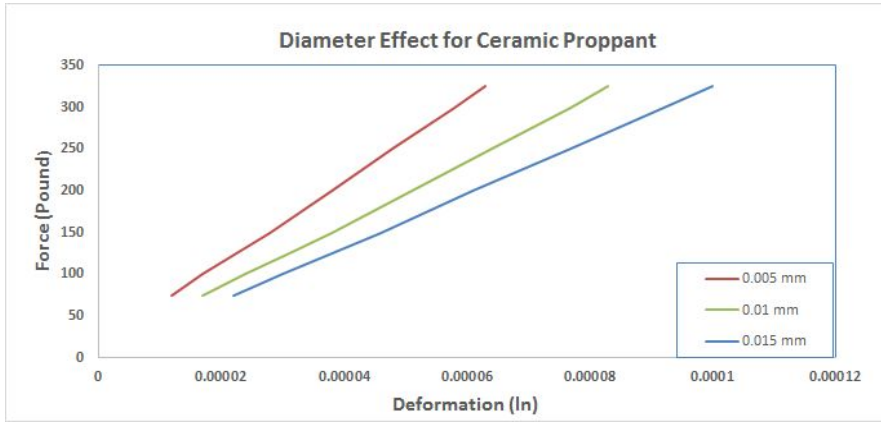


# Application (Results)





# Application (Results)





# Numerical Model (Code)

```
#Ceramic
atom_style granular
atom_modify map array
boundary m m m
newton off
#echo both
communicatesingle vel yes
units s
region reg block -0.005 0.005 -0.0025 0.0025 0.0000 0.008 units box
create_box 1 reg
neighbor 0.002 bin
neigh_modify delay 0
#Material properties required for new pair styles
fix m1 all property/global youngsModulus peratomtype 375.e6
fix m2 all property/global poissonsRatio peratomtype 0.22
fix m3 all property/global coefficientRestitution peratomtypepair 0.3 0.3
fix m4 all property/global coefficientFriction peratomtypepair 0.5 0.5
##New pair style
pair_style gran model hertz tangential history #Hertzian without cohesion
pair_coeff 0.00001
timestep 0.00001
fix xwalls1 all wall/gran model hertz tangential history primitive type 1 xplane -0.0115
fix xwalls2 all wall/gran model hertz tangential history primitive type 1 xplane +0.015
fix ywalls1 all wall/gran model hertz tangential history primitive type 1 yplane -0.015
fix ywalls2 all wall/gran model hertz tangential history primitive type 1 yplane +0.015
fix zwalls1 all wall/gran model hertz tangential history primitive type 1 zplane 0.00
fix zwalls2 all wall/gran model hertz tangential history primitive type 1 zplane 0.03
#servo wall
fix cad all mesh/surface/stress file meshes/lwall.stl type 1 stress on
fix cad2 all mesh/surface/stress/servo file meshes/uwall.stl type 1 com 0. 0. 0. ctrlPV
force axis 0. 0. 0.01 target_val -0.5 vel_max 0.001 kp 2069.
fix geometry all wall/gran model hertz tangential history mesh n_meshes 2 meshes cad
cad2
#distributions for insertion
```

Young's modulus

Poisson's ratio

Friction coefficient

Dimension of figure

```
fix constant 0.015 pts1 all particletemplate/sphere 15485863 atom_type 1 density constant 2500 radius
fix constant 0.015 pts2 all particletemplate/sphere 15485867 atom_type 1 density constant 2500 radius
fix pdd1 all particledistribution/discrete 32452843 2 pts1 0.3 pts2 0.7

#parameters for gradually growing particle diameter
variable alphastart equal 0.25
variable alphatarget equal 0.67
variable growts equal 50000
variable growevery equal 40
variable relaxts equal 20000

#region and insertion
group nve_group region reg

#particle insertion
fix ins nve_group insert/pack seed 32452867 distributiontemplate pdd1 &
maxattemp 200 insert_every once overlapcheck yes all_in yes vel constant 0. 0. 0. &
region reg volumefraction_region 0.9

#apply nve integration to all particles that are inserted as single particles
fix integr nve_group nve/sphere

#output settings, include total thermal energy
#compute 1 all erotate/sphere
#thermo_style custom step atoms ke c_1 vol
#thermo 1000
#thermo_modify lost ignore norm no

#insert the first particles
run 1
dump dmp all custom/vtk 350 post/packing_*.vtk id type type x y z ix iy iz vx vy vz fx fy fz
omegax omegay omegaz radius
dump D_stl all stl 400 post/Wall-*.stl
dump I_stl all mesh/gran/VTK 400 post/Stress_file-*.vtk stress
unfix ins
run 340000
```



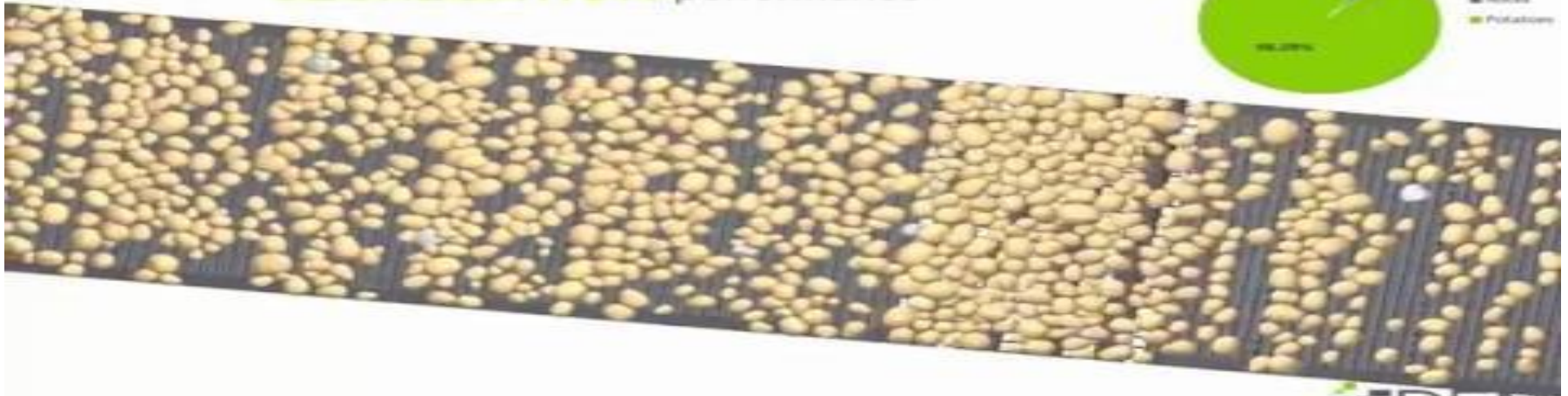
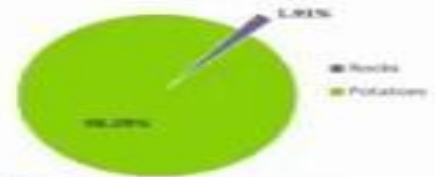


# Applications: Potato Harvest

Time: 41.48 s

Identify rocks and assess  
**SEGREGATION** performance

% by Mass



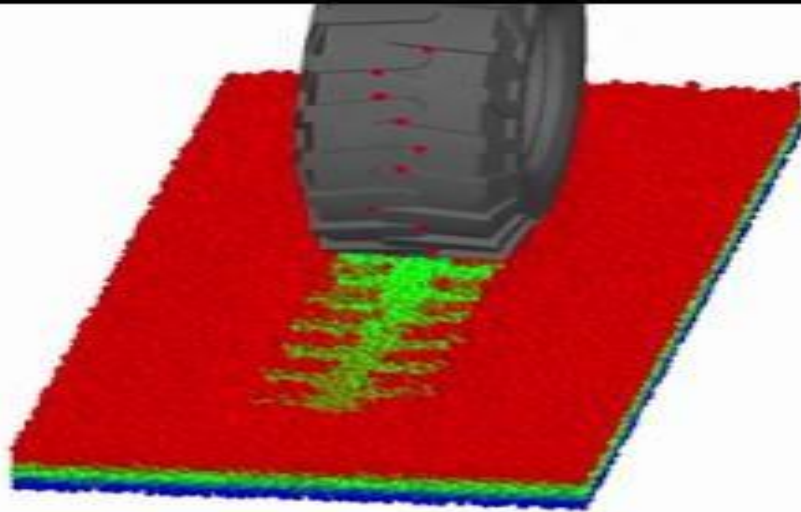
EDEM





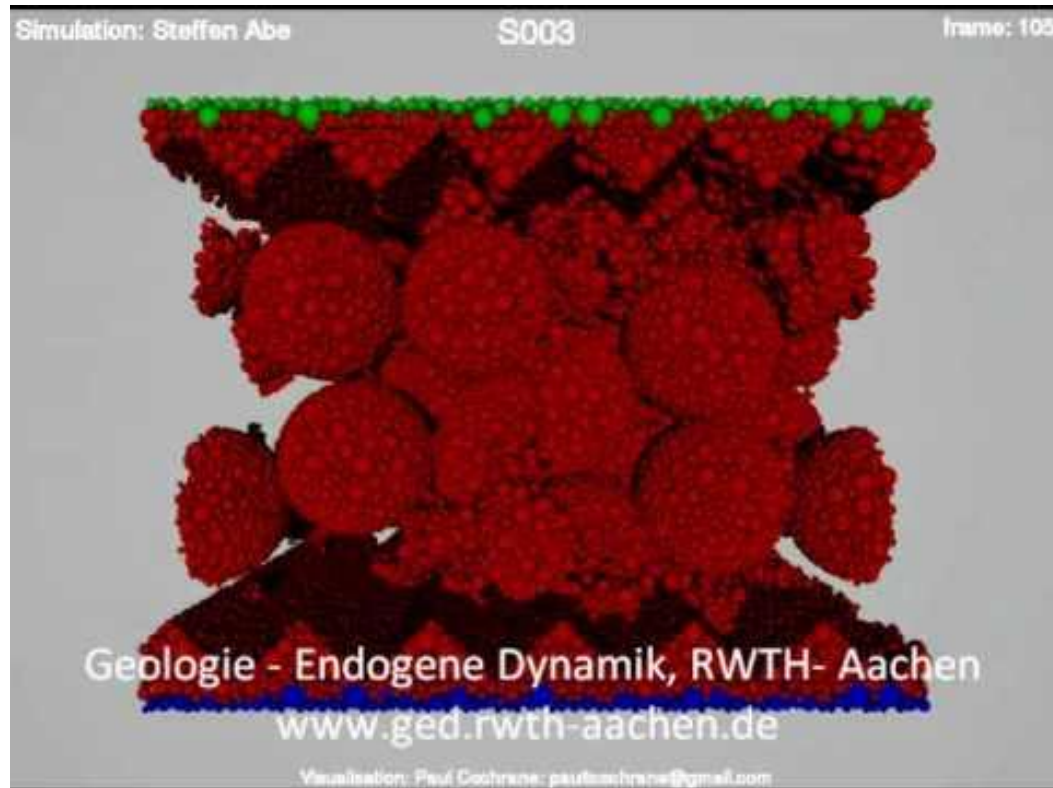
# Applications: Tractor Tire-Soil Interaction

Time: 2.33014 s



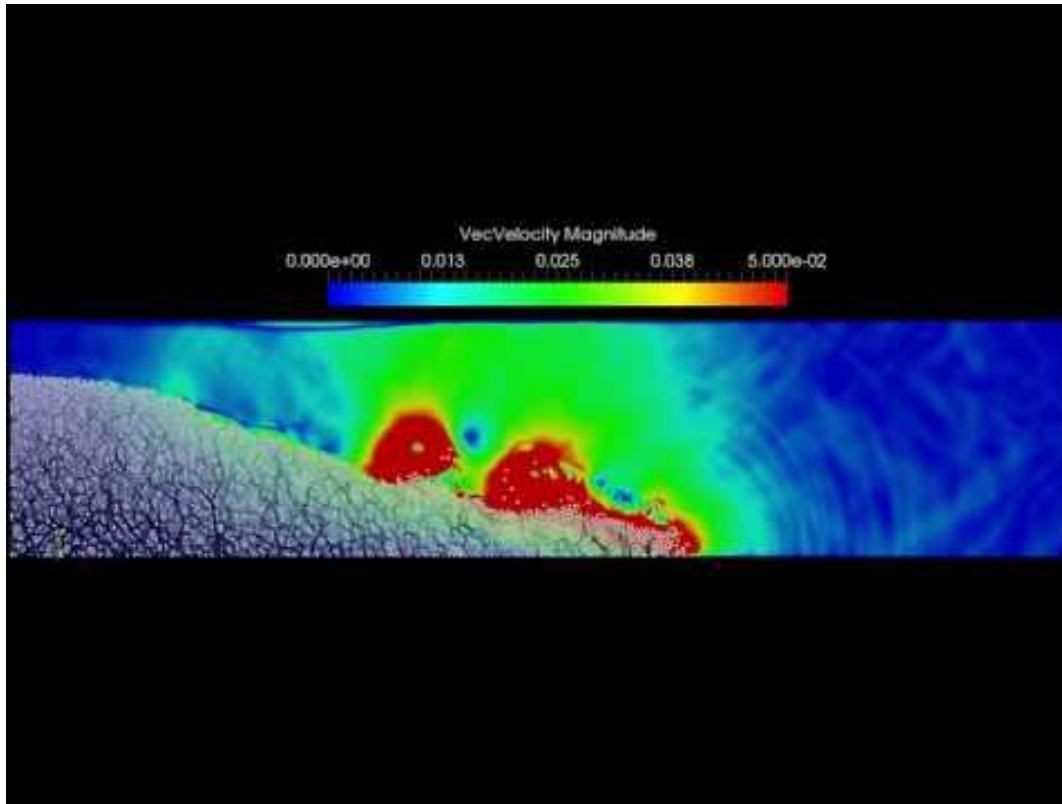


# Applications: Fault Gouge





# Applications: Granular Avalanche





# References

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3. Cundall, P. A.; Strack, O. D. L. (March 1979). "A discrete numerical model for granular assemblies". *Géotechnique*. 29 (1): 47–65. doi:10.1680/geot.1979.29.1.47.
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8. [http://www.sharetechnote.com/html/DE\\_Modeling\\_Example\\_SpringMass.html#SingleSpring\\_SimpleHarmonic\\_Vert\\_Damp](http://www.sharetechnote.com/html/DE_Modeling_Example_SpringMass.html#SingleSpring_SimpleHarmonic_Vert_Damp)
9. <http://hyperphysics.phy-astr.gsu.edu/hbase/oscd.html>
10. <https://ebookcentral.proquest.com/lib/pensu/reader.action?docID=1745057>
11. <http://farside.ph.utexas.edu/teaching/315/Waves/node10.html>