

Convective Heat Transfer in a Geothermal Heat Pump

1. Introduction

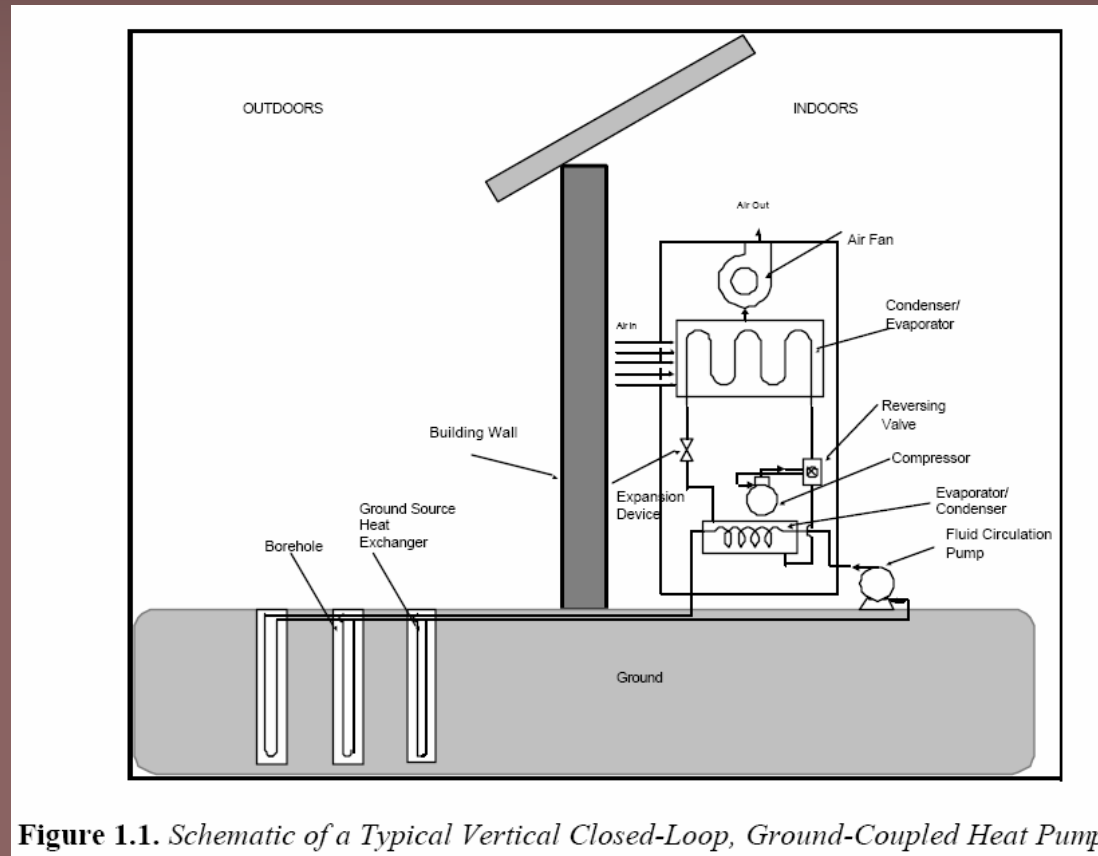


Figure 1.1. Schematic of a Typical Vertical Closed-Loop, Ground-Coupled Heat Pump

Geothermal heat pumps, also called Ground source heat pumps, offer an attractive option for heating and cooling residential and commercial buildings.

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2. Governing Equations

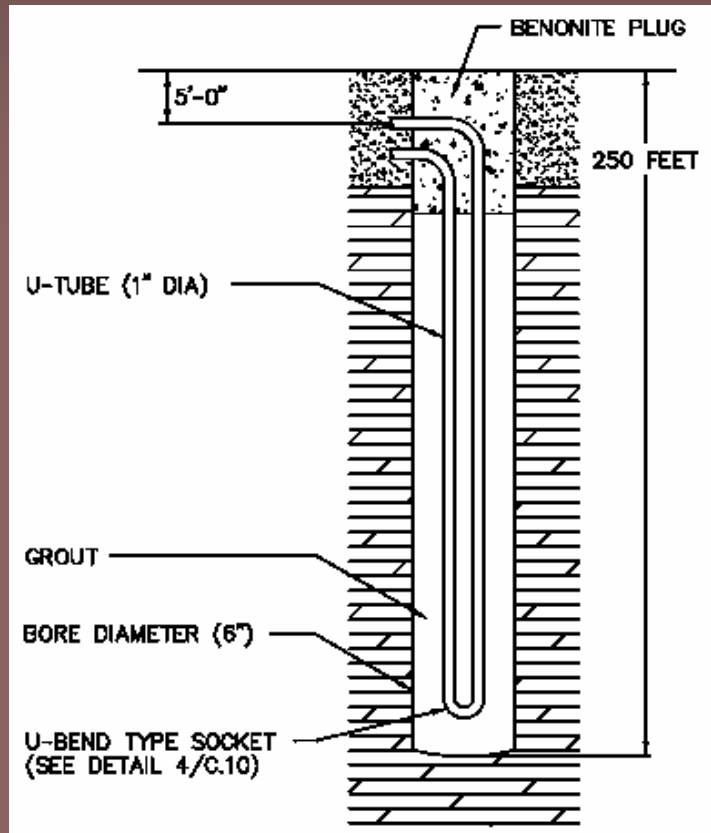


Figure 2-1: Sketch of geothermal heat exchanger.

1 →

$$-\nabla \cdot \eta (\nabla \mathbf{u} + (\nabla \mathbf{u})^T) + \rho (\mathbf{u} \cdot \nabla) \mathbf{u} + \nabla p = 0$$
$$\nabla \cdot \mathbf{u} = 0$$

2 →

$$\nabla \cdot (-k \nabla T + \rho C_p T \mathbf{u}) = 0$$

3 →

$$\mathbf{u} \cdot \mathbf{n} = u_0$$
$$T = T_0$$

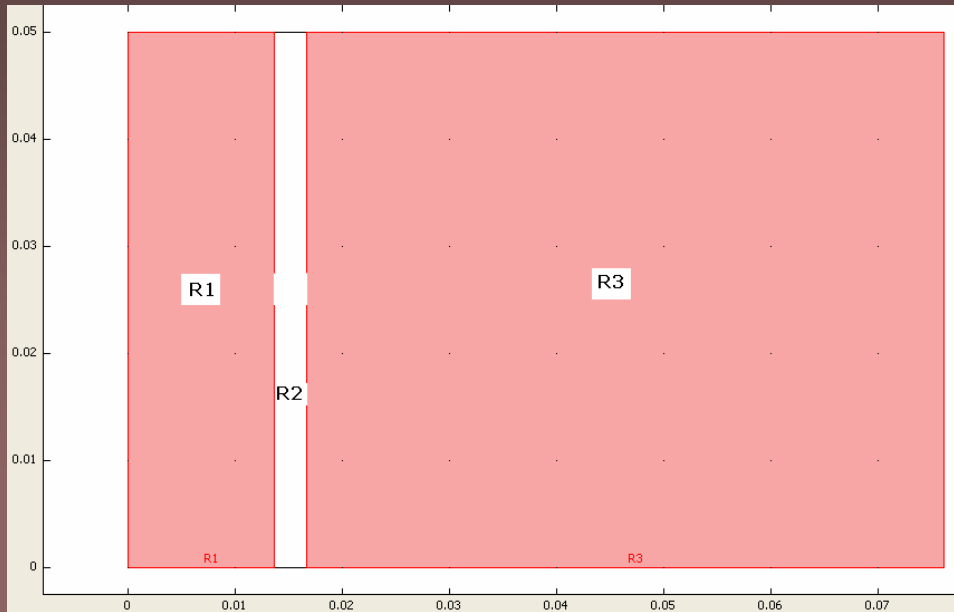
4 →

$$\mathbf{q} \cdot \mathbf{n} = \rho C_p T \mathbf{u} \cdot \mathbf{n}$$

5 →

$$\mathbf{u} = \bar{\mathbf{0}}$$
$$T = T_1$$

3. Formulation



Thermal Properties: **Water**

eta	1.00E-03	Dynamic Viscosity	(kg m ⁻¹ s ⁻¹)
rho	1.00E+03	Density	(kg-3)
Cp	4.20E+03	Heat Capacity	(J kg ⁻¹ K ⁻¹)
k	0.6	Thermal Conductivity	(J s ⁻¹ m ⁻¹ K ⁻¹)
Tin	301.8	Temperature entering	(K)
Tw	284.3	Temperature wall	(K)

Thermal Properties: **PVC**

eta	1.00E+00	Dynamic Viscosity	(kg m ⁻¹ s ⁻¹)
rho	1.76E+03	Density	(kg-3)
Cp	3.20E+02	Heat Capacity	(J kg ⁻¹ K ⁻¹)
k	0.1	Thermal Conductivity	(J s ⁻¹ m ⁻¹ K ⁻¹)

Thermal Properties: **Concrete**

eta	1.00E+00	Dynamic Viscosity	(kg m ⁻¹ s ⁻¹)
rho	2.30E+03	Density	(kg-3)
Cp	1.28E+03	Heat Capacity	(J kg ⁻¹ K ⁻¹)
k	1.8	Thermal Conductivity	(J s ⁻¹ m ⁻¹ K ⁻¹)

Incompressible Navier-Stokes

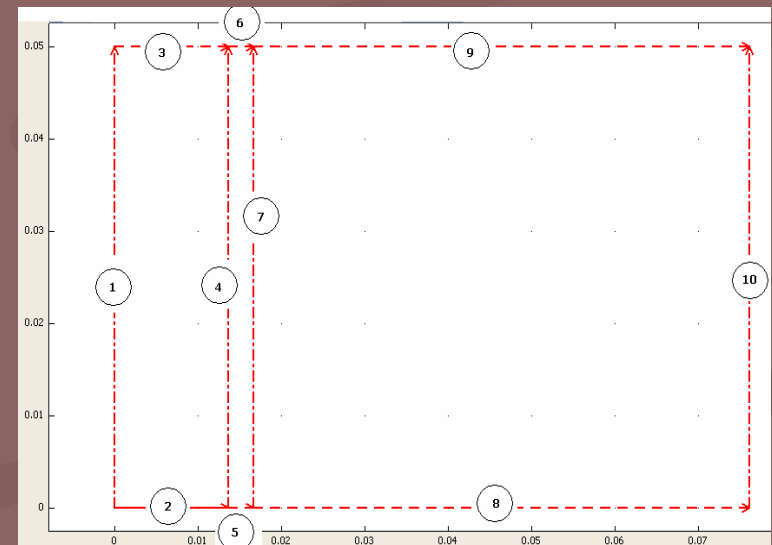
Boundary	1	2	3	4	5	6	7	8	9	10
Type	S/S	I/O	Nf/P	/	NS	Nf/P	/	NS	Nf/P	S/S
u		0								
v		v0								
p			0							

Notes:
 S/S: Slip/Symmetry
 I/O: Inflow/Outflow Velocity
 Nf/P: Normal flow/Pressure
 NS: No-Slip

Convection and Conduction

Boundary	1	2	3	4	5	6	7	8	9	10
Type	TI	T	CF	/	TI	TI	/	TI	TI	T
T		Tin								Tw

Notes:
 TI: Thermal Insulation
 T: Temperature
 HF: Heat Flux
 CF: Convective Flux



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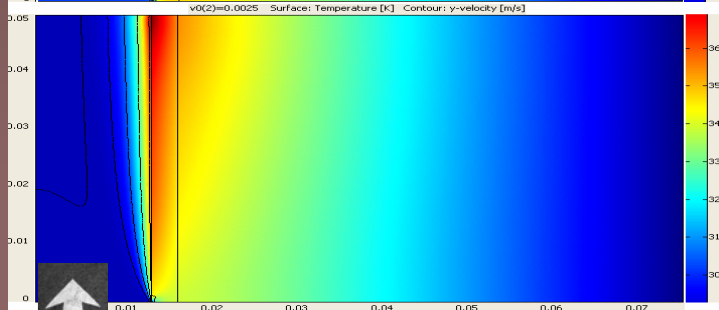
4. Solution

From top to bottom:
Convection decreases as
velocity increase.

.0005
→
(m/s)



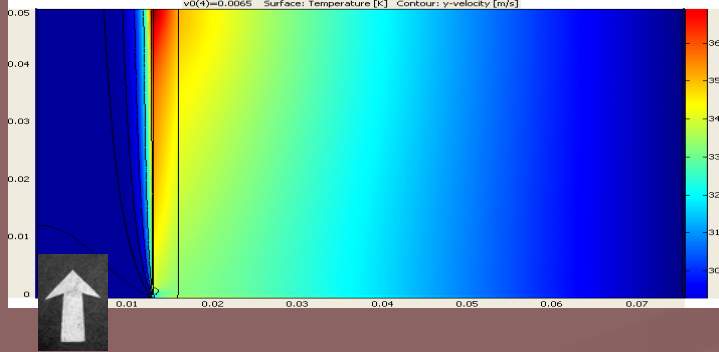
.0025
→
(m/s)



.0045
→
(m/s)

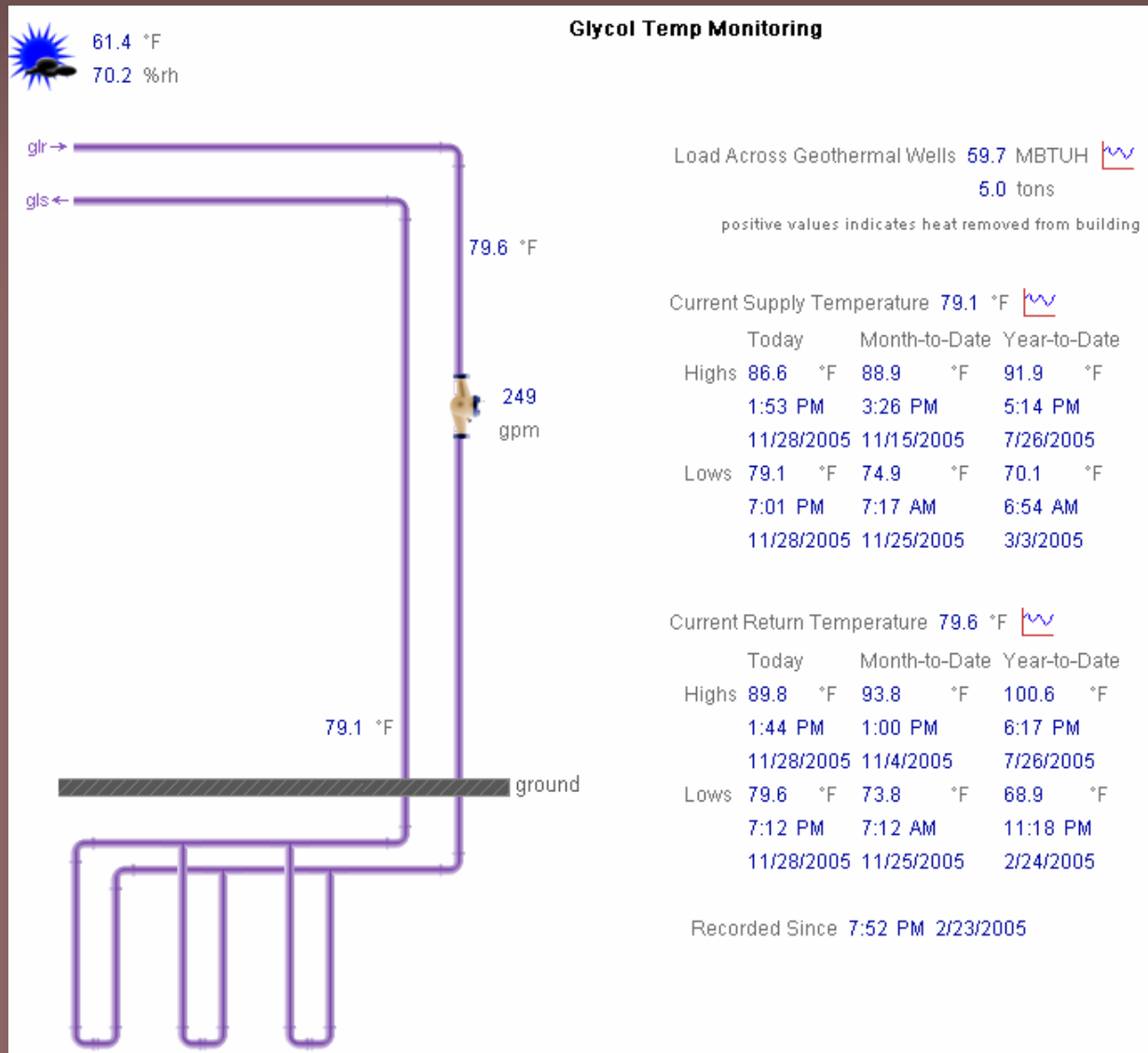


.0065
→
(m/s)



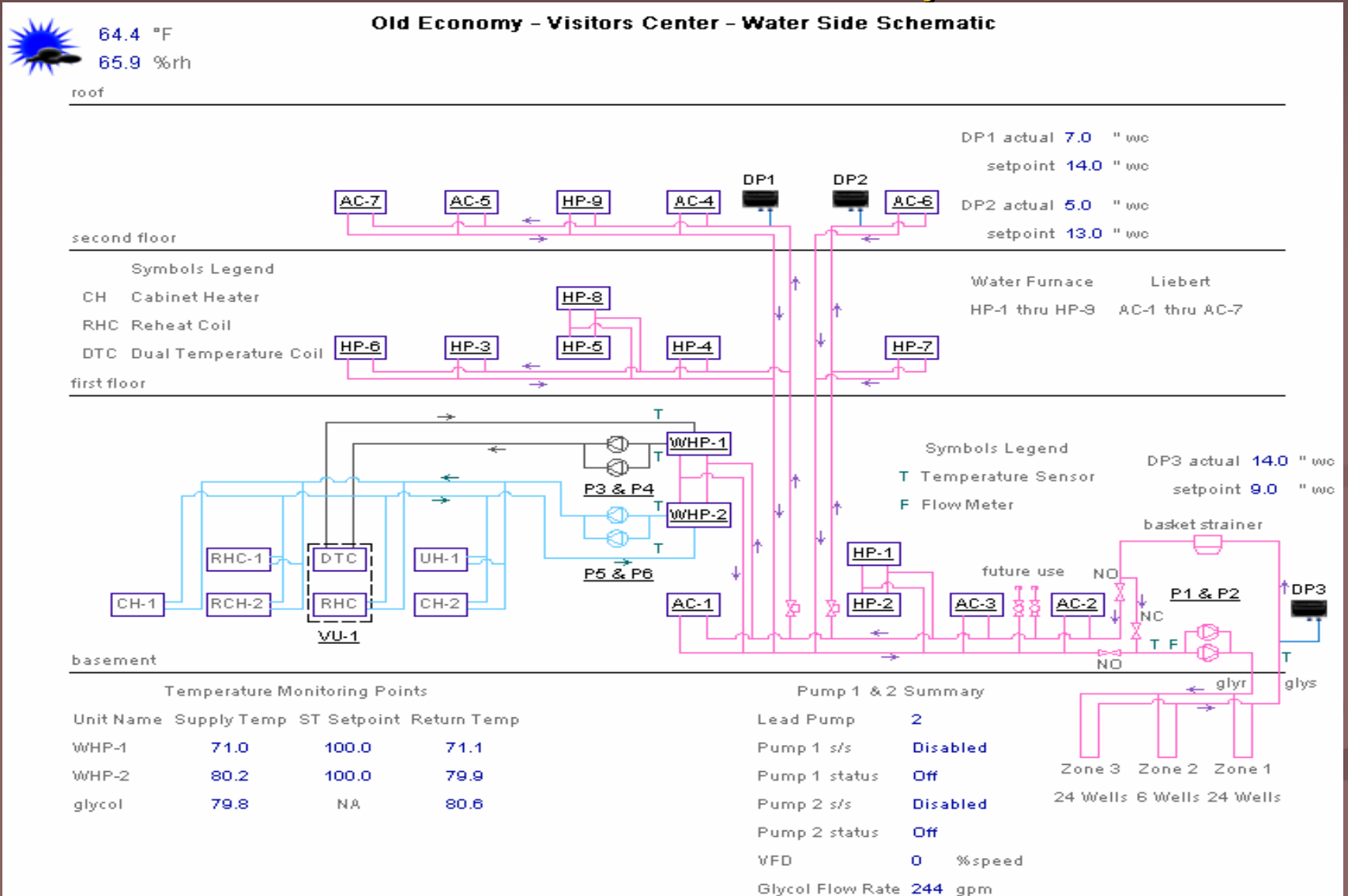
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5. Validation



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6. Parametric Study



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7. Conclusions

1. Varying flow has more impact in ground source heat transfer systems than varying temperature (T_{in})
2. Flow is varied in real systems using variable speed drives coupled to the building load profile through building automation control systems.
3. Ability to visualize remote or hidden processes lead to better understanding of systems.